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<https://hdl.handle.net/2324/4055221>

出版情報 : Ocean Engineering. 184, pp.11-22, 2019-07-15. Elsevier

バージョン :

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<https://doi.org/10.1016/j.oceaneng.2019.05.018>

Received 15 January 2019,

Revised 1 May 2019,

Accepted 8 May 2019,

Available online 17 May 2019.

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Abstract: A semi-analytic model based on the linear potential theory is developed to solve wave radiation from an array of truncated cylinders with arbitrary cross sections, which are free to oscillate independently in the water of finite depth. The radiated velocity potential in the interior region below each cylinder and an exterior region extending to the infinity are expressed as a series of eigenfunctions in the local cylindrical coordinate system using the method of separation of variables. They are solved subsequently by imposing the continuity condition of the velocity and the pressure at the wetted surfaces of the cylinders and the fluid interfaces between adjacent regions. In the process, the Fourier series expansion method is applied to the terms regarding the arbitrary cross sections. The model is successfully validated through a case study with two caissons, and an excellent agreement is obtained with a numerical model based on the boundary element method. The semi-analytic model is then applied to explore the influence of the layout (angle and spacing between the two caissons) on the added-mass and radiation damping coefficients. It is found that the hydrodynamic interaction between the two caissons can increase or decrease the value of the coefficients depending on the wave frequency.

Keywords: Wave radiation; Floating cylinder; Hydrodynamic coefficients; Added mass; Radiation damping; Potential flow.

1. Introduction

Floating structures undergoing harmonic oscillations in still water produce radiated waves; the oscillations cause the water to react on the structure, resulting in hydrodynamic forces – the so-called wave radiation forces. It is well known that the response of marine structures can be significantly influenced by the radiation forces. Therefore, solving the problem of wave radiation from offshore structures has been a subject of considerable research interest (Bai and Eatock Taylor, 2006; Krishnankutty and Vendhan, 1995; Newman, 2005; Sheng et al., 2015; Taghipour et al., 2008).

The radiation forces exerted on structures oscillating harmonically are generally split into two terms: one, in phase with the velocity of the structure; the other, with its acceleration. Hydrodynamic coefficients – more specifically, radiation damping and added-masses – are introduced to express the radiation forces. To solve the radiation problem of a truncated circular cylinder oscillating time-harmonically in finite-depth water, Yeung (1981) divided the water domain into an interior region below the cylinder and an exterior region extending to the infinity in the horizontal plane. The velocity potential in each region was expressed as a series of eigenfunctions in a cylindrical coordinate system. The hydrodynamic coefficients, including coupling coefficients for sway and roll, were obtained analytically. A similar analytical method was applied by Bhatta and Rahman (2003) and Bhatta (2007) to compute the added-mass and damping coefficients in the case of a heaving circular cylinder, for various depth-to-radius and draft-to-radius ratios. For a given depth-to-radius ratio, the heave added-mass was found to increase with the draft-to-radius ratio. Jiang et al. (2014) proposed analytical solutions for the wave radiation by a submerged circular cylinder, and found that the heave and pitch added-masses of the cylinder can be twice as large as that of itself. Circular cylinders in other situations, e.g., floating in extremely shallow water or in front of

a vertical wall, have also been studied (Drobyshevski, 2004; Teng et al., 2004; Yu and Srivastava, 2012; Zheng and Zhang, 2015, 2016).

Wave radiation from an elliptical cylinder either submerged and resting on the seabed or partially immersed was analytically investigated by Williams and Darwiche (1990). As in Yeung (1981), the fluid was divided into interior and exterior regions; in contrast, the governing equation and boundary conditions were written in elliptical coordinates, and the velocity potentials in each region were expressed in terms of infinite series involving Mathieu and modified Mathieu functions. This technique was later adopted by Zhang and Williams (1996) in solving the wave radiation problem from a submerged elliptical disk. Hydrodynamic properties of the disk were found to be strongly dependent on both the aspect ratio and relative submergence. More recently, Yu et al. (2019) proposed an analytical model that can be used to deal with the problem of radiation from a floating cylinder with an arbitrary cross section. The Fourier series method was adopted to describe the wetted surface body conditions and continuity conditions between adjacent regions.

In the case of an array of cylinders – ubiquitous in ocean energy and offshore engineering applications, e.g., offshore wind farms and wave farms – the oscillation of the cylinders in unison or independently from one another will give rise to hydrodynamic interactions between them, further complicating the radiation problem. Kagemoto and Yue (1986) developed an interaction theory which can be used to predict the wave hydrodynamic coefficients given only the diffraction characteristics of individual members. Following Linton and Evans (1990), who had applied a direct method to the wave diffraction at multiple circular uniform cylinders, Kim (1993) studied the wave radiation for six-degree-of-freedom motions of N bottom-mounted vertical circular cylinders and obtained the radiation potentials as well as the added-mass and wave damping coefficients in closed forms. Yilmaz and Incecik (1998) employed Kagemoto and Yue (1986)'s method to calculate the hydrodynamic coefficients of an array of truncated circular cylinders oscillating as a whole. Therefore, his model cannot be used to solve the radiation problem of an array of cylinders if they oscillate individually, as is the case in a floating offshore wind farm or a wave farm. To overcome this limitation, Siddorn and Eatock Taylor (2008) considered wave diffraction and radiation by multiple truncated circular cylinders which are free to oscillate independently. Extensive results were presented for the hydrodynamic coefficients of a group of four truncated circular cylinders free to respond independently in translational modes, i.e., surge, sway and heave. The analytical results of the hydrodynamic coefficients in rotational modes, e.g., roll and pitch, can be found in Zeng and Tang (2013). More recently, Zheng and Zhang (2018) and Zheng et al. (2018) investigated the wave radiation from an array of truncated circular cylinders with one or some of them hollowed out. Concerning non-circular cylinders, the hydrodynamic problems of arrays of elliptical cylinders subjected to linear regular waves were studied analytically by Chatjigeorgiou and Mavrakos (2010) and Chatjigeorgiou (2013), considering fixed cylinders, therefore with no wave radiation forces.

The objective of this paper is to investigate wave radiation from multiple truncated cylinders with arbitrary cross sections and free to oscillate independently from one another, considering the hydrodynamic interactions between them – a problem that has not been addressed so far. This is carried out by means of an ad hoc semi-analytic model, which is used to evaluate the added-mass and wave radiation damping coefficients, together with the hydrodynamic interaction coefficients. The semi-analytic model constitutes an option to deal with wave radiation from multiple vertical cylinders with arbitrary cross sections without the utilization of any BEM-based numerical software.

2. Mathematical model

An array of truncated cylinders of arbitrary cross sections floating in water with a depth of h (Fig. 1) is considered. The cylinders are numbered $n = 1, 2, \dots, N$, and located in a Cartesian coordinate system $Oxyz$ with the plane $z = 0$ on the mean water level (MWL) and the Oz axis pointing vertically upwards. For Cylinder n , a cylindrical coordinate system denoted as $O_n r_n \theta_n z$ is adopted, with the origin O_n set within the cross sections of the cylinder at MWL, with coordinates $(x_n, y_n, 0)$ in the $Oxyz$ system. A point defined as (x_n, y_n, z_n) in the $Oxyz$ system is adopted as the reference point of the rotational modes of Cylinder n to consider the radiation problem induced by the rotational motions. The submerged depth of each cylinder can be different, denoted as d_n for Cylinder n , respectively.

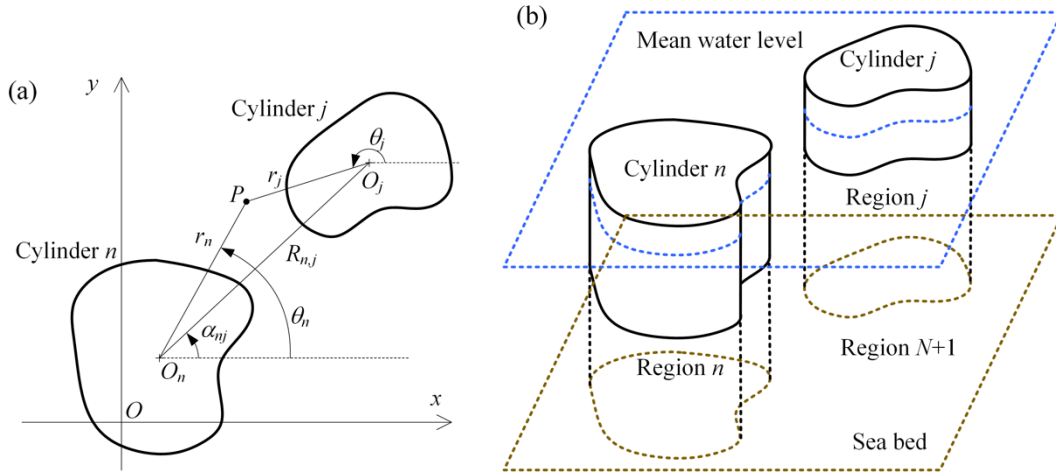


Fig. 1. Schematic of the truncated cylinders of arbitrary cross sections: a) plan view; b). side view

To describe the shape of the cross section of Cylinder n , a function denoted as S_n is introduced in the $O_n r_n \theta_n z$ system as:

$$S_n(r_n, \theta_n) = r_n - R_n(\theta_n), \quad (1)$$

where $R_n(\theta_n)$ denotes the radius of any point at the edge of the cross section at θ_n . Therefore, the normal unit vector pointing into the water at the side surface of Cylinder n can be written in the $O_n r_n \theta_n z$ system as:

$$\vec{n} = \frac{1}{\sqrt{1 + \left(\frac{1}{r_n} \frac{\partial S_n}{\partial \theta_n} \right)^2}} \left(\vec{e}_{r,n} + \frac{1}{r_n} \frac{\partial S_n}{\partial \theta_n} \vec{e}_{\theta,n} + 0 \vec{e}_z \right), \quad (2)$$

in which $(\vec{e}_{r,n}, \vec{e}_{\theta,n}, \vec{e}_z)$ are the unit basis vectors in the $O_n r_n \theta_n z$ system.

For a cylinder with the cross-section edge varying sharply at some points, as in the case of rectangular and triangular cylinders, the normal unit vector at the side surface might be discontinuous at these points. For such cases, a series of Fourier functions can be adopted to represent R_n and S_n , following Liu et al. (2017).

As shown in Fig. 1b, the entire water domain is divided into N interior regions in which Region n ($n=1, 2, \dots, N$) is the “projection” region below Cylinder n , bounded by the cylinder bottom, seabed and the extension of the cylinder side surface, and a single exterior region, i.e.,

Region $N+1$, which circumscribes the cylinders and their “projections” and extends to the horizontal infinity.

It is assumed that water is inviscid and incompressible, the flow is irrotational, and each of the cylinders oscillates harmonically with a small amplitude; therefore, the linear potential theory may be used to investigate the radiation problem from multiple cylinders. When the multiple cylinders oscillate, radiated waves are generated, and the total radiated velocity potential can be expressed as

$$\phi(x, y, z, t) = \text{Re} \left[\Phi(x, y, z) e^{-i\omega t} \right], \quad (3)$$

where t is the time, Φ is a complex spatial velocity potential independent of time, $i = \sqrt{-1}$, and ω is the angular frequency of the harmonic oscillation.

The spatial velocity potential Φ may be expressed as a superposition (linear combination) of radiated potentials caused by each individual cylinder oscillating in six modes of oscillation:

$$\Phi = \sum_{n=1}^N \sum_{i=1}^6 \Phi_R^{n,i} u^{n,i}, \quad (4)$$

where $\Phi_R^{n,i}$ represents the spatial velocity potential due to a unit-amplitude velocity oscillation of Cylinder n in i -th mode ($i=1,2,\dots,6$ denoting the modes of surge, sway, heave, roll, pitch, and yaw, respectively), and $u^{n,i}$ denotes the corresponding complex velocity amplitude.

It follows that $\Phi_R^{n,i}$ must satisfy the three-dimensional Laplace's equation in the entire water domain,

$$\frac{\partial^2 \Phi_R^{n,i}}{\partial x^2} + \frac{\partial^2 \Phi_R^{n,i}}{\partial y^2} + \frac{\partial^2 \Phi_R^{n,i}}{\partial z^2} = 0, \quad (5)$$

the kinematical condition on the seabed,

$$\frac{\partial \Phi_R^{n,i}}{\partial z} = 0, \quad z = -h, \quad (6)$$

should also be satisfied, as well as the linearized free surface condition,

$$\frac{\partial \Phi_R^{n,i}}{\partial z} - \frac{\omega^2}{g} \Phi_R^{n,i} = 0, \quad z = 0 \quad \text{and} \quad S_j \geq 0. \quad (7)$$

The boundary conditions at the wetted surface of all cylinders that $\Phi_R^{n,i}$ must satisfy can be written as:

$$\frac{\partial \Phi_R^{n,i}}{\partial z} = \delta_{n,j} (\delta_{3,i} + \delta_{4,i} r_j \sin \theta_j - \delta_{5,i} r_j \cos \theta_j), \quad z = -d_j \quad \text{and} \quad S_j < 0, \quad (8)$$

$$r_j^2 \frac{\partial \Phi_R^{n,i}}{\partial r_j} + \frac{\partial S_j}{\partial \theta_j} \frac{\partial \Phi_R^{n,i}}{\partial \theta_j} = \delta_{n,j} f_1^{n,i}, \quad -d_j \leq z \leq 0 \quad \text{and} \quad S_j = 0, \quad (9)$$

where $\delta_{i,j}$ represents the Kronecker delta function, and

$$f_1^{n,i} = \begin{cases} r_n^2 \left(\cos \theta_n - \frac{\partial S_n}{r_n \partial \theta_n} \sin \theta_n \right), & i = 1 \\ r_n^2 \left(\sin \theta_n + \frac{\partial S_n}{r_n \partial \theta_n} \cos \theta_n \right), & i = 2 \\ 0, & i = 3 \\ -(z - z_n) r_j^2 \left(\sin \theta_n + \frac{\partial S_n}{r_n \partial \theta_n} \cos \theta_n \right), & i = 4 \\ (z - z_n) r_n^2 \left(\cos \theta_n - \frac{\partial S_n}{r_n \partial \theta_n} \sin \theta_n \right), & i = 5 \\ r_n^2 \frac{\partial S_n}{\partial \theta_n}, & i = 6 \end{cases}. \quad (10)$$

The radiation condition at infinity is:

$$\sqrt{k_0 r_n} \left(\frac{\partial \Phi_{\mathbf{R}}^{n,i}}{\partial r_n} - i k_0 \Phi_{\mathbf{R}}^{n,i} \right) = 0, \quad S_j \rightarrow \infty, \quad (11)$$

where k_0 is the wave number, which satisfies the dispersion relation, i.e.,

$$\omega^2 = g k_0 \tanh(k_0 h). \quad (12)$$

3 The solution to radiated potentials

3.1 Radiated spatial potentials in different regions

For interior and exterior regions, i.e., Region j ($j=1,2, \dots, N$) and Region $N+1$, as divided in Fig. 1b, the radiated spatial potential can be expressed in different forms. The spatial velocity potentials $\Phi_{\mathbf{R}}^{n,i}$ in Region j , denoted as $\Phi_{\mathbf{R},j}^{n,i}$, are given as follows:

1) Region j ($j=1,2, \dots, N$)

$$\Phi_{\mathbf{R},j}^{n,i}(r_j, \theta_j, z) = \delta_{n,j} \Phi_{\mathbf{R},p}^{n,i} + \sum_{m=-\infty}^{\infty} \left[A_{m,0}^{n,i,j} (r_j)^{|m|} + \sum_{l=1}^{\infty} A_{m,l}^{n,i,j} I_m(\beta_{j,l} r_j) \cos[\beta_{j,l} (z+h)] \right] e^{im\theta_j}, \quad (13)$$

where $A_{m,l}^{n,i,j}$ are the unknown coefficients to be determined; I_m is the modified Bessel function of first kind and order m ; $\beta_{j,l}$ is the eigenvalue, which is given by

$$\beta_{j,l} = \frac{l\pi}{h - d_j}, \quad l=0, 1, 2, 3, \dots, \quad (14)$$

$\Phi_{\mathbf{R},p}^{n,i}$ is a particular solution, which can be expressed in the $O_n r_n \theta_n z$ coordinate system as

$$\Phi_{R,p}^{n,i} = \begin{cases} 0, & i=1,2,6 \\ \frac{1}{4(h-d_n)} \left[2(z+h)^2 - r_n^2 \right], & i=3 \\ \frac{\sin \theta_n}{8(h-d_n)} \left[4r_n(z+h)^2 - r_n^3 \right], & i=4 \\ \frac{\cos \theta_n}{8(h-d_n)} \left[r_n^3 - 4r_n(z+h)^2 \right], & i=5 \end{cases} \quad (15)$$

2) Region $N+1$

In Region $N+1$, the radiated spatial potential can be decomposed into the summation of N cylindrical spatial potentials radiated from N cylinders:

$$\Phi_{R,N+1}^{n,i} = \sum_{j=1}^N \Phi_j^{n,i,e}, \quad (16)$$

where $\Phi_j^{n,i,e}$ represents the wave traveling outwards from Cylinder j , which can be written in terms of the following well-known eigenfunction expansion in its cylindrical coordinate system:

$$\Phi_j^{n,i,e}(r_j, \theta_j, z) = \sum_{m=-\infty}^{\infty} \left[B_{m,0}^{n,i,j} H_m(k_0 r_j) \frac{Z_0(z)}{Z_0(0)} + \sum_{l=1}^{\infty} B_{m,l}^{n,i,j} K_m(k_l r_j) \frac{Z_l(z)}{Z_l(0)} \right] e^{im\theta_j}, \quad (17)$$

in which

$$Z_0(z) = N_0^{-1/2} \cosh[k_0(z+h)]; \quad Z_l(z) = N_l^{-1/2} \cos[k_l(z+h)]; \quad (18)$$

$$N_0 = \frac{1}{2} \left[1 + \frac{\sinh(2k_0 h)}{2k_0 h} \right]; \quad N_l = \frac{1}{2} \left[1 + \frac{\sin(2k_l h)}{2k_l h} \right]; \quad (19)$$

$B_{m,0}^{n,i,j}$ are the coefficients to be solved in Section 3.2; H_m is the Hankel function of the first kind of order m ; K_m is the modified Bessel function of the second kind of order m ; k_l is the eigenvalue, which is given by

$$\omega^2 = -k_l g \tan(k_l h), \quad l=1,2,3, \dots \quad (20)$$

By using the Graf's addition theorem for Bessel functions (Abramowitz and Stegun, 1964; Zheng and Zhang, 2015, 2016, 2018), the expression of $\Phi_{R,N+1}^{n,i,e}$ can be converted from its own $O_n r_n \cdot \theta_n \cdot z$ system into the $O_j r_j \cdot \theta_j \cdot z$ system ($n \neq j$) for $r_j \leq R_{jn'}$, and $\Phi_{R,N+1}^{n,i}$ in the $O_j r_j \cdot \theta_j \cdot z$ system can be expressed by:

$$\begin{aligned} \Phi_{R,N+1}^{n,i}(r_j, \theta_j, z) = & \sum_{m=-\infty}^{\infty} \left[B_{m,0}^{n,i,j} H_m(k_0 r_j) \frac{Z_0(z)}{Z_0(0)} + \sum_{l=1}^{\infty} B_{m,l}^{n,i,j} K_m(k_l r_j) \frac{Z_l(z)}{Z_l(0)} \right] e^{im\theta_j} \\ & + \sum_{\substack{n'=1 \\ n' \neq j}}^N \sum_{m=-\infty}^{\infty} \left\{ B_{m,0}^{n,i,n'} \frac{Z_0(z)}{Z_0(0)} \sum_{m'=-\infty}^{\infty} (-1)^{m'} H_{m-m'}(k_0 R_{n'j}) J_{m'}(k_0 r_j) e^{i(m\alpha_{nj} - m'\alpha_{jn'})} e^{im'\theta_j} \right. \\ & \left. + \sum_{l=1}^{\infty} B_{m,l}^{n,i,n'} \frac{Z_l(z)}{Z_l(0)} \sum_{m'=-\infty}^{\infty} K_{m-m'}(k_l R_{n'j}) I_{m'}(k_l r_j) e^{i(m\alpha_{nj} - m'\alpha_{jn'})} e^{im'\theta_j} \right\} \end{aligned} \quad (21)$$

where J_m is the Bessel function of order m .

3.2 Computation method for the coefficients

Eqs. (13) and (21) for the radiated spatial potentials given in Sections 3.1 must also satisfy the boundary conditions, Eq. (9), and the continuity conditions on the interfaces of any two adjacent regions, i.e., $S_j=0$ ($j=1,2, \dots, N$). These conditions can be combined as follows:

1) Continuity of pressure at the boundary $S_j=0$:

$$\Phi_{R,N+1}^{n,i} \Big|_{S_j=0} = \Phi_{R,j}^{n,i} \Big|_{S_j=0}, \quad -h < z < -d_j, \quad (22)$$

2) Continuity of normal velocity at the boundary $S_j=0$:

$$\left(r_j^2 \frac{\partial_{R,N+1}^{n,i}}{\partial r_j} + \frac{\partial S_j}{\partial \theta_j} \frac{\partial_{R,N+1}^{n,i}}{\partial \theta_j} \right) \Big|_{S_j=0} = \begin{cases} \delta_{n,j} f_1^{n,i} \Big|_{S_j=0}, & -d_j < z < 0 \\ \left(r_j^2 \frac{\partial \Phi_{R,j}^{n,i}}{\partial r_j} + \frac{\partial S_j}{\partial \theta_j} \frac{\partial \Phi_{R,j}^{n,i}}{\partial \theta_j} \right) \Big|_{S_j=0}, & -h < z < -d_j \end{cases}. \quad (23)$$

After inserting Eqs. (13) and (21) into Eqs. (22) and (23), it is readily observed that the terms obtained regarding r_n and $\partial S_n / \partial \theta_n$ are both dependent on θ_n at $S_n=0$. Fourier series can be employed to express these items as follows:

$$J_m(k_0 r_n) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,0,q}^{J,n} e^{iq\theta_n}; \quad \left(r_n^2 k_0 J'_m(k_0 r_n) + im J_m(k_0 r_n) \frac{\partial S_n}{\partial \theta_n} \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,0,q}^{J',n} e^{iq\theta_n}, \quad (24a)$$

$$H_m(k_0 r_n) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,0,q}^{H,n} e^{iq\theta_n}; \quad \left(r_n^2 k_0 H'_m(k_0 r_n) + im H_m(k_0 r_n) \frac{\partial S_n}{\partial \theta_n} \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,0,q}^{H',n} e^{iq\theta_n}, \quad (24b)$$

$$K_m(k_l r_n) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,l,q}^{K,n} e^{iq\theta_n}; \quad \left(r_n^2 k_l K'_m(k_l r_n) + im K_m(k_l r_n) \frac{\partial S_n}{\partial \theta_n} \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,l,q}^{K',n} e^{iq\theta_n}, \quad (24c)$$

$$I_m(k_l r_n) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,l,q}^{I,n} e^{iq\theta_n}; \quad \left(r_n^2 k_l I'_m(k_l r_n) + im I_m(k_l r_n) \frac{\partial S_n}{\partial \theta_n} \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,l,q}^{I',n} e^{iq\theta_n}, \quad (24d)$$

$$I_m(\beta_{n,l} r_n) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} \tilde{f}_{m,l,q}^{I,n} e^{iq\theta_n}; \quad \left(r_n^2 \beta_{n,l} I'_m(\beta_{n,l} r_n) + im I_m(\beta_{n,l} r_n) \frac{\partial S_n}{\partial \theta_n} \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} \tilde{f}_{m,l,q}^{I',n} e^{iq\theta_n}, \quad (24e)$$

$$(r_n)^{|m|} \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_{m,0,q}^{R,n} e^{iq\theta_n}; \quad \left(|m|(r_n)^{|m|+1} + im(r_n)^{|m|} \frac{\partial S_n}{\partial \theta_n} \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} \tilde{f}_{m,0,q}^{R,n} e^{iq\theta_n}, \quad (24f)$$

in which, for convenience, the Fourier coefficients on the right-hand side of Eqs. (24a)~(24f) can be represented by $\Re_{n,q}$ and evaluated from

$$\Re_{n,q} = \frac{1}{2\pi} \int_{-\pi}^{\pi} \Psi_{n,q}(\theta_n) e^{-iq\theta_n} d\theta_n, \quad (25)$$

where $\Psi_{n,q}$ denotes the θ_n dependent items as given on the left-hand side of Eqs. (24a)~(24f).

Multiplying both sides of Eq. (22) by $e^{-i\tau\theta_j} \cos[\beta_{j,\xi}(z+h)]/(h-d_j)$, integrating for $\theta_j \in [-\pi, \pi]$ and $z \in [-h, -d_j]$, and making some rearrangements, we obtain, for any pair of integer (τ, ξ) :

$$1 \quad - \sum_{m=-\infty}^{\infty} A_{m,\tau,\xi}^{1,j} A_{m,\xi}^{n,i,j} + \sum_{m=-\infty}^{\infty} \sum_{l=0}^{\infty} B_{m,l}^{n,i,j} L_{l,\xi}^{(j)} A_{m,\tau,l}^{2,j} + \sum_{\substack{n'=1 \\ n' \neq j}}^N \sum_{m=-\infty}^{\infty} \sum_{l=0}^{\infty} B_{m,l}^{n,i,n'} L_{l,\xi}^{(j)} T_{m,\tau,l}^{j,n'} = \delta_{n,j} f_{\tau,\xi}^{(1),n,i}, \quad (26)$$

2 where

$$3 \quad A_{m,\tau,\xi}^{1,n} = \begin{cases} f_{m,\xi,\tau-m}^{R,n}, & \xi = 0 \\ \frac{1}{2} \tilde{f}_{m,\xi,\tau-m}^{I,n}, & \xi = 1, 2, 3, \dots \end{cases}, \quad A_{m,\tau,l}^{2,n} = \begin{cases} f_{m,0,\tau-m}^{H,n}, & l = 0 \\ f_{m,l,\tau-m}^{K,n}, & l = 1, 2, 3, \dots \end{cases}, \quad (27)$$

$$4 \quad L_{l,\xi}^{(n)} = \frac{1}{h-d_n} \int_{-h}^{-d_n} \frac{Z_l(z) \cos[\beta_{n,\xi}(z+h)]}{Z_l(0)} dz$$

$$= \begin{cases} \frac{(-1)^\xi (h-d_n) k_0 \sinh[k_0(h-d_n)]}{[(h-d_n)^2 k_0^2 + \xi^2 \pi^2] \cosh(k_0 h)} & l = 0; \xi = 0, 1, 2, \dots \\ \frac{(-1)^\xi (h-d_n) k_l \sin[k_l(h-d_n)]}{[(h-d_n)^2 k_l^2 - \xi^2 \pi^2] \cos(k_l h)} & l = 1, 2, 3, \dots; \xi = 0, 1, 2, \dots \end{cases}, \quad (28)$$

$$5 \quad T_{m,\tau,l}^{n,j} = \begin{cases} \sum_{m'=-\infty}^{\infty} (-1)^{m'} H_{m-m'}(k_0 R_{jn}) f_{m',0,\tau-m}^{J,n} e^{i(m\alpha_{jn}-m'\alpha_{nj})}, & l = 0 \\ \sum_{m'=-\infty}^{\infty} K_{m-m'}(k_l R_{jn}) f_{m',l,\tau-m}^{I,n} e^{i(m\alpha_{jn}-m'\alpha_{nj})}, & l = 1, 2, 3, \dots \end{cases}, \quad (29)$$

$$6 \quad f_{\tau,\xi}^{(1),n,i} = \begin{cases} 0, & i = 1, 2, 6 \\ \frac{1}{4(h-d_n)^2} [2\delta_{\tau,0} IZ_{n,\xi} - \delta_{\xi,0} (h-d_n) f_{2,0,\tau}^{R,n}], & i = 3 \\ \frac{1}{16i(h-d_n)^2} [4IZ_{n,\xi} (f_{1,0,\tau-1}^{R,n} - f_{1,0,\tau+1}^{R,n}) - \delta_{\xi,0} (h-d_n) (f_{3,0,\tau-1}^{R,n} - f_{3,0,\tau+1}^{R,n})], & i = 4 \\ \frac{-1}{16(h-d_n)^2} [4IZ_{n,\xi} (f_{1,0,\tau-1}^{R,n} + f_{1,0,\tau+1}^{R,n}) - \delta_{\xi,0} (h-d_n) (f_{3,0,\tau-1}^{R,n} + f_{3,0,\tau+1}^{R,n})], & i = 5 \end{cases} \quad (30)$$

7 in which

$$8 \quad IZ_{n,\xi} = \int_{-h}^{-d_n} \cos[\beta_{n,\xi}(z+h)] (z+h)^2 dz = \begin{cases} \frac{(h-d_n)^3}{3}, & \xi = 0 \\ \frac{2(-1)^\xi (h-d_n)^3}{\xi^2 \pi^2}, & \xi = 1, 2, 3, \dots \end{cases} \quad (31)$$

9 After multiplying both sides of Eq.(23) by $e^{-i\tau\theta_j} Z_\xi(z)/h$ and integrating for $\theta_j \in [-\pi, \pi]$ and z
10 $\in [-h, 0]$, and making some rearrangements, we obtain, for any pair of integer (τ, ξ) :

$$11 \quad -\frac{h-d_j}{h} Z_\xi(0) \sum_{m=-\infty}^{\infty} \sum_{l=0}^{\infty} A_{m,l}^{n,i,j} L_{\xi,l}^{(j)} A_{m,\tau,l}^{3,j} + \sum_{m=-\infty}^{\infty} B_{m,\xi}^{n,i,j} A_{m,\tau,\xi}^{4,j} + \sum_{\substack{n'=1 \\ n' \neq j}}^N \sum_{m=-\infty}^{\infty} B_{m,\xi}^{n,i,n'} T_{m,\tau,\xi}^{j,n'} = \delta_{n,j} f_{\tau,\xi}^{(2),n,i}, \quad (32)$$

12 where

$$A_{m,\tau,l}^{3,n} = \begin{cases} \tilde{f}_{m,0,\tau-m}^{R,n}, & l=0 \\ \tilde{f}_{m,l,\tau-m}^{I',n}, & l=1,2,3,\dots \end{cases}, \quad A_{m,\tau,\xi}^{4,n} = \begin{cases} \frac{f_{m,0,\tau-m}^{H',n}}{Z_0(0)}, & \xi=0 \\ \frac{f_{m,\xi,\tau-m}^{K',n}}{Z_\xi(0)}, & \xi=1,2,3,\dots \end{cases}, \quad (33)$$

$$T_{m,\tau,l}^{n,j} = \begin{cases} \frac{1}{Z_0(0)} \sum_{m'=-\infty}^{\infty} (-1)^{m'} H_{m-m'}(k_0 R_{jn}) f_{m',0,\tau-m}^{J',n} e^{i(m\alpha_{jn}-m'\alpha_{nj})}, & l=0 \\ \frac{1}{Z_l(0)} \sum_{m'=-\infty}^{\infty} K_{m-m'}(k_l R_{jn}) f_{m',l,\tau-m}^{I',n} e^{i(m\alpha_{jn}-m'\alpha_{nj})}, & l=1,2,3,\dots \end{cases}. \quad (34)$$

Since the expression for $f_{\tau,\xi}^{(2),n,i}$ is rather complicated, it is given in Appendix A.

Eqs. (26) and (32) can be seen as a linear algebraic system for the unknown coefficients $A_{m,l}^{n,i,j}$ and $B_{m,l}^{n,i,j}$, which can be solved after truncation. We truncate $(2M+1)$ terms ($m=-M, \dots, 0, \dots, M$) and $(L+1)$ terms ($l=0, 1, \dots, L$) in Eqs. (13) and (21), and take $(\tau=-M, \dots, 0, \dots, M)$ and $(\xi=0, 1, \dots, L)$ in Eqs. (26) and (32), with the result that a linear algebraic system of $2N(2M+1)(L+1)$ order is obtained with the same number of unknown coefficients, which can be solved easily. In the semi-analytic computations below, $M=15$ and $L=8$ are found to yield accurate results.

4 Hydrodynamic coefficients

The radiation force/moment exerted on Cylinder n' in Mode i' due to a unit-amplitude velocity oscillation of Cylinder n oscillating in Mode i is $\text{Re}[F_{n,i}^{n',i'} e^{-i\omega t}]$, in which $F_{n,i}^{n',i'}$ is the complex amplitude of the force/moment. Since the oscillation of cylinders is harmonic, $F_{n,i}^{n',i'}$ can be generally written in terms of hydrodynamic coefficients as:

$$F_{n,i}^{n',i'} = -i\omega\rho \int_{S_{n'}} \Phi_{\mathbf{R}}^{n,i} n_i' ds = i\omega a_{n,i}^{n',i'} - c_{n,i}^{n',i'}, \quad (35)$$

where the hydrodynamic coefficients $a_{n,i}^{n',i'}$ and $c_{n,i}^{n',i'}$ are the so-called added-mass and radiation damping coefficients, respectively; $S_{n'}$ is the wetted surface of Cylinder n' ; n_i' is the generalized normal, with $n_1=n_x$, $n_2=n_y$, $n_3=n_z$, $n_4=-(z-z_n)n_y+(y-y_n)n_z$, $n_5=(z-z_n)n_x-(x-x_n)n_z$, $n_6=-(y-y_n)n_x+(x-x_n)n_y$; and $\vec{n} = n_x\vec{i} + n_y\vec{j} + n_z\vec{k}$ is the normal unit vector at the cylinder surface considered, Eq. (2), which may be written in the $Oxyz$ system as

$$\vec{n} = \frac{1}{\sqrt{1 + \left(\frac{1}{r_n} \frac{\partial S_n}{\partial \theta_n}\right)^2}} \left(\left(\cos \theta_n - \frac{1}{r_n} \frac{\partial S_n}{\partial \theta_n} \sin \theta_n \right) \vec{i} + \left(\sin \theta_n + \frac{1}{r_n} \frac{\partial S_n}{\partial \theta_n} \cos \theta_n \right) \vec{j} + 0\vec{k} \right), \quad (36)$$

in which $(\vec{i}, \vec{j}, \vec{k})$ are the unit basis vectors in the $Oxyz$ system.

5 Results and discussion

5.1 Model validation

The example used for validation of the present semi-analytic model is two caissons deployed in parallel (Fig. 2), representative of mobile offshore bases (Lamas-Pardo et al., 2015) and offshore floating bridges (Eidem, 2017).

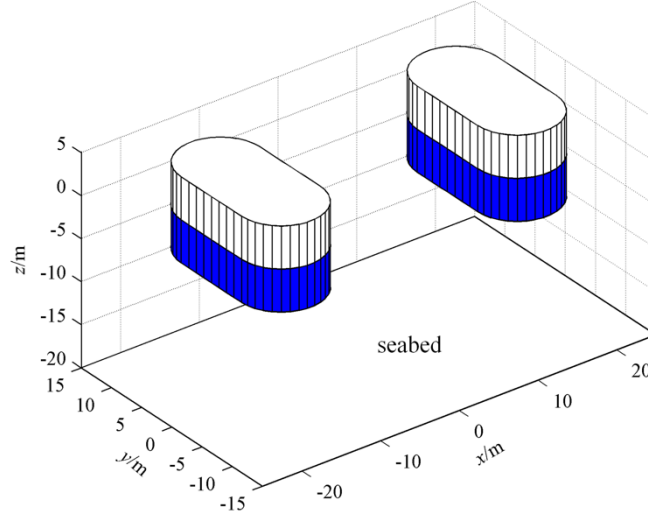


Fig. 2. Case study for validation: two parallel caissons.

In the present case study, the caissons have the same dimensions. The cross section of each caisson consists of a central rectangle of 10×10 m and two external half circles with a diameter of 10 m. Hence, the functions S_n and $\partial S_n / \partial \theta_n$ of the cross section for each caisson in its cylindrical coordinate system can be written as

$$S_n = r_n \begin{cases} \frac{5}{\cos \theta_n}, & |\theta_n| \leq \frac{\pi}{4} \\ \frac{-5}{\cos \theta_n}, & |\theta_n - \pi| \leq \frac{\pi}{4} \\ 10|\sin \theta_n|, & \text{others} \end{cases}, \quad \frac{\partial S_n}{\partial \theta_n} = \begin{cases} \frac{-5 \tan \theta_n}{\cos \theta_n}, & |\theta_n| \leq \frac{\pi}{4} \\ \frac{5 \tan \theta_n}{\cos \theta_n}, & |\theta_n - \pi| \leq \frac{\pi}{4} \\ -10 \cos \theta_n, & \frac{\pi}{4} \leq \theta_n \leq \frac{3\pi}{4} \\ 10 \cos \theta_n, & -\frac{3\pi}{4} \leq \theta_n \leq -\frac{\pi}{4} \end{cases}. \quad (37)$$

The spacing between the caissons is $e = 20$ m. The immersed depth of each caisson and the water depth are $d_n = 5$ m and $h = 20$ m, respectively.

The results may be normalized by means of the nondimensional hydrodynamic parameters:

$$\bar{a}_{n,i}^{n',i'} = \frac{a_{n,i}^{n',i'}}{\rho h^j}, \quad \bar{c}_{n,i}^{n',i'} = \frac{c_{n,i}^{n',i'}}{\omega \rho h^j}, \quad (38)$$

where $j=3$ for $i=1,2,3$ and $i'=1,2,3$; $j=4$ for $i=1,2,3$ and $i'=4,5,6$; $j=4$ also applies to $i=4,5,6$ and $i'=1,2,3$; whereas $j=5$ for $i=4,5,6$ and $i'=4,5,6$.

To validate the present semi-analytic model, the hydrodynamic coefficients of the case study (Fig. 2) are evaluated by using both the present semi-analytic model and a BEM-based numerical software, ANSYS-AQWA (ANSYS-AQWA, 2011). Both the semi-analytic results and the numerical results of the frequency responses of the added-mass and radiation damping coefficients for $e/h=1.0$ are plotted in Figs. 3 and 4, in which the frequency is normalized as kh (hereafter, k is adopted as a simplified form of k_0). Figs. 3a and 4a illustrate the hydrodynamic coefficients of Caisson 1 regarding the translational modes of oscillation, i.e., surge, sway and heave. The hydrodynamic coefficients of Caisson 1 regarding the rotational modes, i.e., roll, pitch and yaw, are

given in Figs. 3b and 4b. The results in Figs. 3c and 4c reflect the hydrodynamic interaction between the two caissons in translational modes. Similarly, the hydrodynamic coefficients regarding the hydrodynamic interaction between the two caissons in rotational modes are presented in Figs. 3d and 4d.

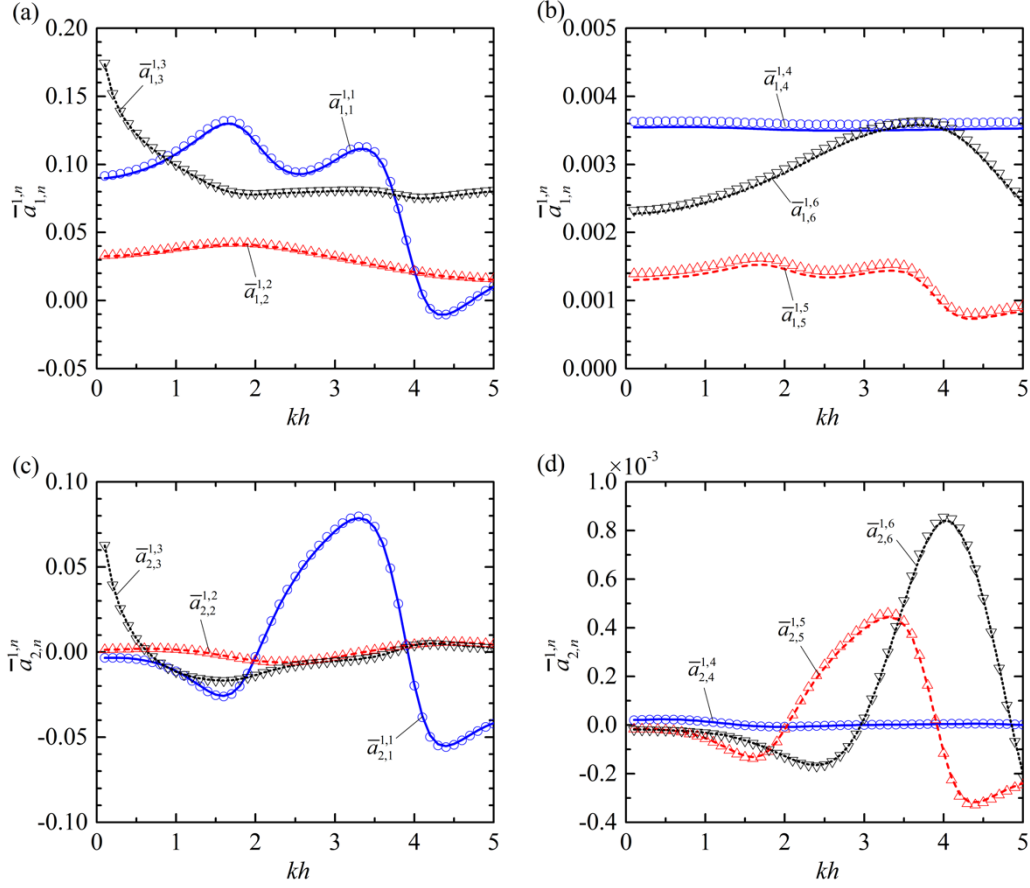


Fig. 3. Added-mass coefficient of the two caissons for $e/h=1.0$: present semi-analytic results (lines) vs. numerical results (symbols): (a) $\bar{a}_{1,n}^{1,n}$, $n=1,2,3$; (b) $\bar{a}_{1,n}^{1,n}$, $n=4,5,6$; (c) $\bar{a}_{2,n}^{1,n}$, $n=1,2,3$; (d) $\bar{a}_{2,n}^{1,n}$, $n=4,5,6$.

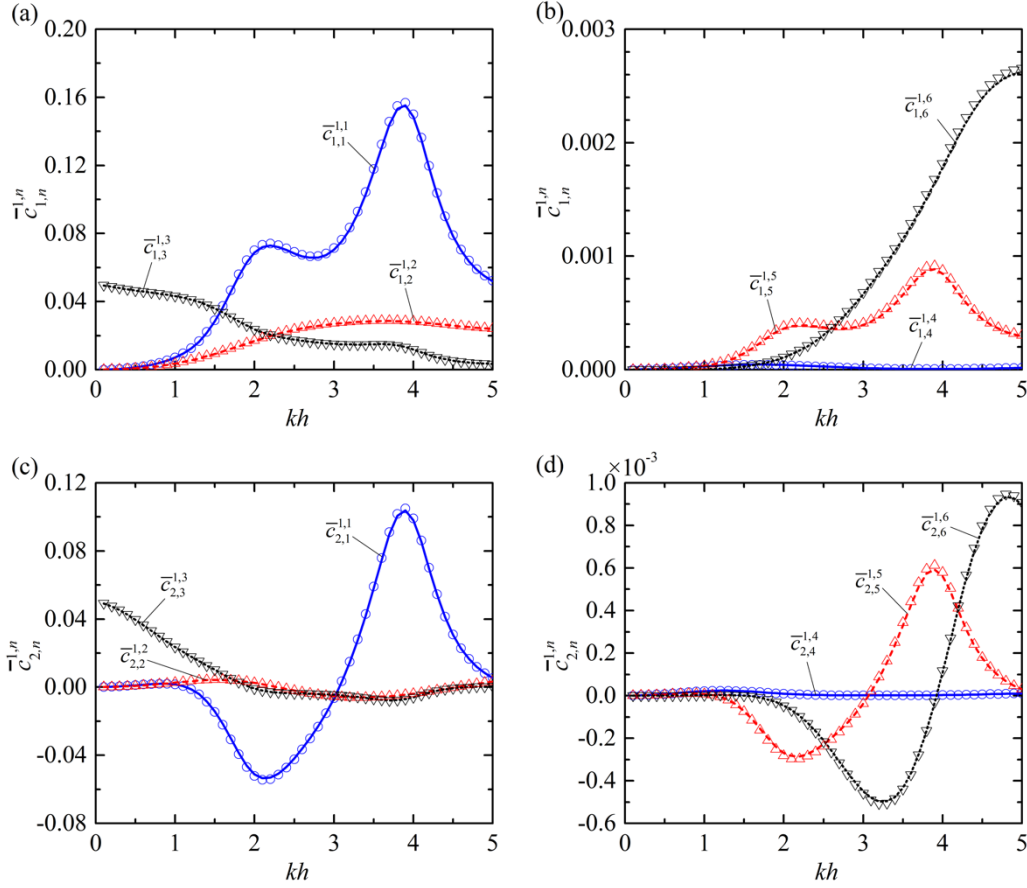


Fig. 4. Radiation damping of the two caissons for $e/h=1.0$: present semi-analytic results (lines) vs. numerical results (symbols): (a) $\bar{c}_{1,n}^{1,n}$, $n=1,2,3$; (b) $\bar{c}_{1,n}^{1,n}$, $n=4,5,6$; (c) $\bar{c}_{2,n}^{1,n}$, $n=1,2,3$; (d) $\bar{c}_{2,n}^{1,n}$, $n=4,5,6$.

The semi-analytic results and the numerical results in terms of both the added-mass/radiation damping of Caisson 1 and the hydrodynamic coefficients reflecting the hydrodynamic interaction between the two caissons are found to agree with each other. The excellent agreement corroborates the correctness of the semi-analytic model.

In the following subsections, the validated semi-analytic model is employed to study the effect of the spacing distance and the deployment angle on hydrodynamic coefficients.

5.2 Effect of the spacing between the caissons

The frequency response of the normalised hydrodynamic coefficients regarding the translational and rotational modes for different spacings, from $e/h=0.5$ to 2.0 , are plotted in Figs. 5 and 6, respectively. Apart from the results of two-caisson cases, hydrodynamic coefficients of an individual caisson floating in isolation are also presented as a comparison. This is equivalent to the two-caisson situation when e/h is large enough to ignore the interaction between the caissons.

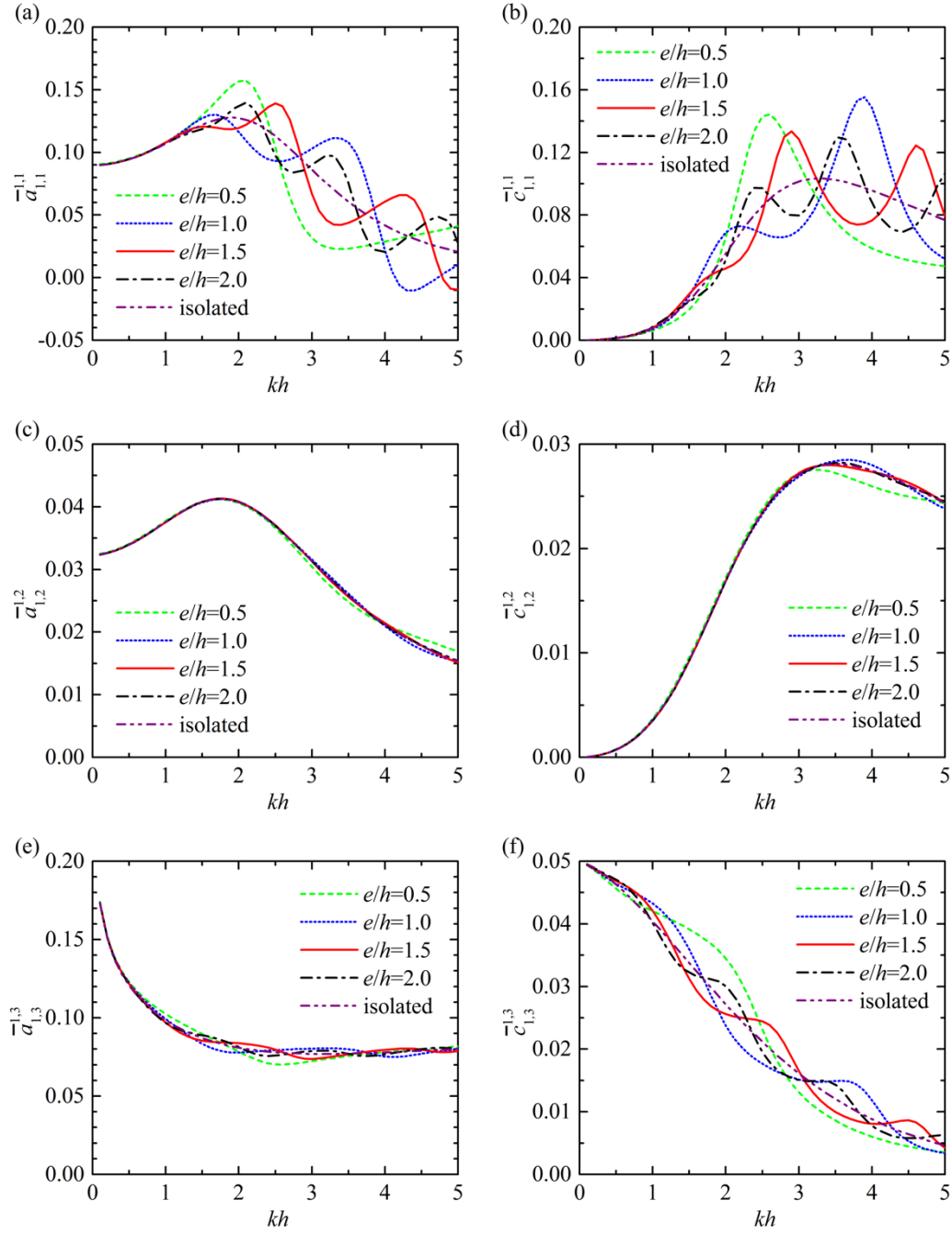
Since the two caissons have the same dimensions and are placed in parallel, it follows that $a_{1,n}^{1,n} = a_{2,n}^{2,n}$, $c_{1,n}^{1,n} = c_{2,n}^{2,n}$. For this reason, the following discussion is concerned with Caisson 1.

Figures 5 and 6 show that $\bar{a}_{1,n}^{1,n}$ and $\bar{c}_{1,n}^{1,n}$ ($n=1,3,5$) are strongly affected by the spacing between the caissons. The curves representing the frequency responses of these hydrodynamic coefficients oscillate around the curve of the isolated caisson case. This is due to the effects of the hydrodynamic interaction for different kh . When Caisson 1 oscillates, apart from the waves radiated

from the caisson itself, Caisson 1 is also subjected to the waves reflected from Caisson 2, i.e., the radiation force can be divided into two components: the waves radiated from Caisson 1 and those reflected from Caisson 2. The frequency of the reflected waves will change with the oscillating frequency of Caisson 1. Consequently, the propagation time of the reflected waves through the spacing between the two caissons will change, and the hydrodynamic force acting on Caisson 1 due to the reflected waves will change as well in terms of both amplitude and phase. When the two components of the radiation force induced by radiated and reflected waves are in close phase, the hydrodynamic coefficients will be strengthened; whereas if they are out of phase, the hydrodynamic coefficients will be weakened. Take $\bar{a}_{1,1}^{1,1}$ of the two caissons case with $e/h=0.5$ as an example, the peak value of $\bar{a}_{1,1}^{1,1}$ for such case reaches 0.157 at $kh=2.1$, where $\bar{a}_{1,1}^{1,1}$ for the isolated caisson case is only 0.126, i.e., for $kh=2.1$, $\bar{a}_{1,1}^{1,1}$ of the two caissons case with $e/h=0.5$ is 0.25 times larger than that of the isolated caisson. The smallest value of $\bar{a}_{1,1}^{1,1}$ of the two caissons case with $e/h=0.5$ occurs at $kh=3.4$, being 0.023, which is 0.65 times smaller than that (0.065) when the caissons are isolated. It can be learned from Fig. 5b that the peak value of $\bar{c}_{1,1}^{1,1}$ with $e/h=0.5$ occurs at $kh=2.6$, which is obviously different from that where the peak of $\bar{a}_{1,1}^{1,1}-kh$ happens.

For the two caissons with a larger value of e and kh ranging from 0 to 5.0, the hydrodynamic coefficient curve of the two caissons oscillates around that of the isolated caisson more frequently, and the oscillating amplitude decreases. This can also be explained from wave reflection from Caisson 2. A larger value of e means a larger propagating distance of the reflected waves, thus for the same variation of kh , the phase of the radiation force component induced by the reflected waves changes more, i.e., the strengthening and weakening effects of the hydrodynamic interaction switch more frequently. As e increases, the reflected waves reaching Caisson 1 decrease due to the conservation of wave energy, resulting in weaker hydrodynamic interaction, i.e., smaller oscillating amplitude. Note that for $e/h=1.0$ and 1.5, negative values of $\bar{a}_{1,1}^{1,1}$ occur around $kh=4.4$ and 5.0, respectively, as a result of the hydrodynamic interaction.

The remaining hydrodynamic coefficients, i.e., $\bar{a}_{1,n}^{1,n}$ and $\bar{c}_{1,n}^{1,n}$ ($n=2,4,6$), are found to be rather insensitive to the spacing between the caissons, especially $\bar{a}_{1,4}^{1,4}$ (Figs. 5 and 6).



1

2 Fig. 5. Normalised hydrodynamic coefficients regarding translational modes vs. kh for four spacings:3 $e/h = 0.5, 1.0, 1.5$ and 2.0 , together with the isolated caisson case: (a) $\bar{a}_{1,1}^{1,1}$; (b) $\bar{c}_{1,1}^{1,1}$; (c) $\bar{a}_{1,2}^{1,2}$; (d)4 $\bar{c}_{1,2}^{1,2}$; (e) $\bar{a}_{1,3}^{1,3}$; (f) $\bar{c}_{1,3}^{1,3}$.

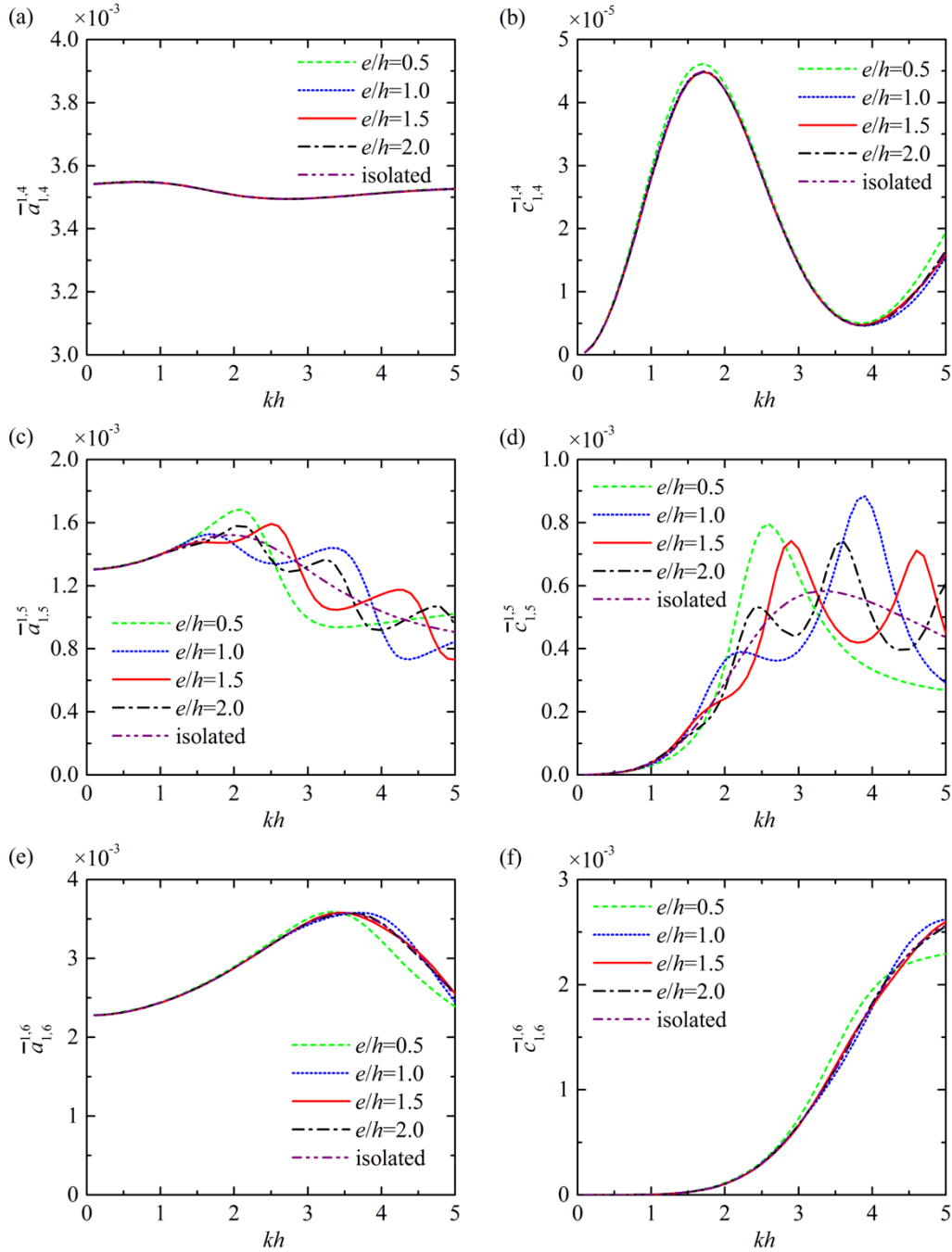


Fig. 6. Normalised hydrodynamic coefficients regarding rotational modes vs kh for four spacings: $e/h = 0.5, 1.0, 1.5$ and 2.0 , together with the isolated caisson case: (a) $\bar{a}_{1,4}^{1,4}$; (b) $\bar{c}_{1,4}^{1,4}$; (c) $\bar{a}_{1,5}^{1,5}$; (d) $\bar{c}_{1,5}^{1,5}$; (e) $\bar{a}_{1,6}^{1,6}$; (f) $\bar{c}_{1,6}^{1,6}$.

5.3 Effect of the deployment angle

During the construction process of mobile offshore bases and the installation of the floating foundations in floating bridges, one caisson could be dragged around another. Hence, the wave radiation problem of the caissons in different configurations deserves consideration, especially when they are in proximity to each other. This subsection deals with two caissons with the same dimensions as in the previous section in different configurations, i.e., with variant values of the

1 deployment angle α (Fig.7).

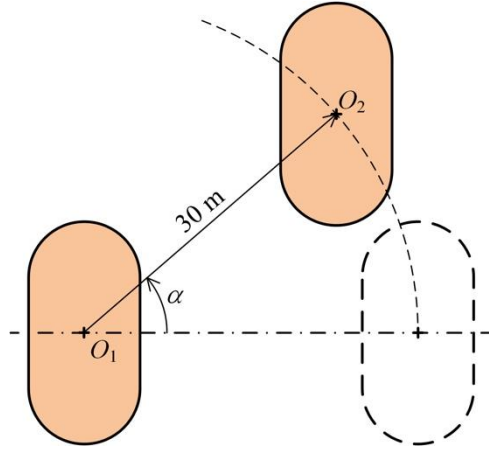


Fig. 7. Definition sketch of the deployment angle

Here the radiation problems from five cases with $\alpha=0, \pi/6, \pi/4, \pi/3$, and $\pi/2$ are analytically solved. Results of the hydrodynamic coefficients for different cases and their comparison are given in Figs. 8 and 9.

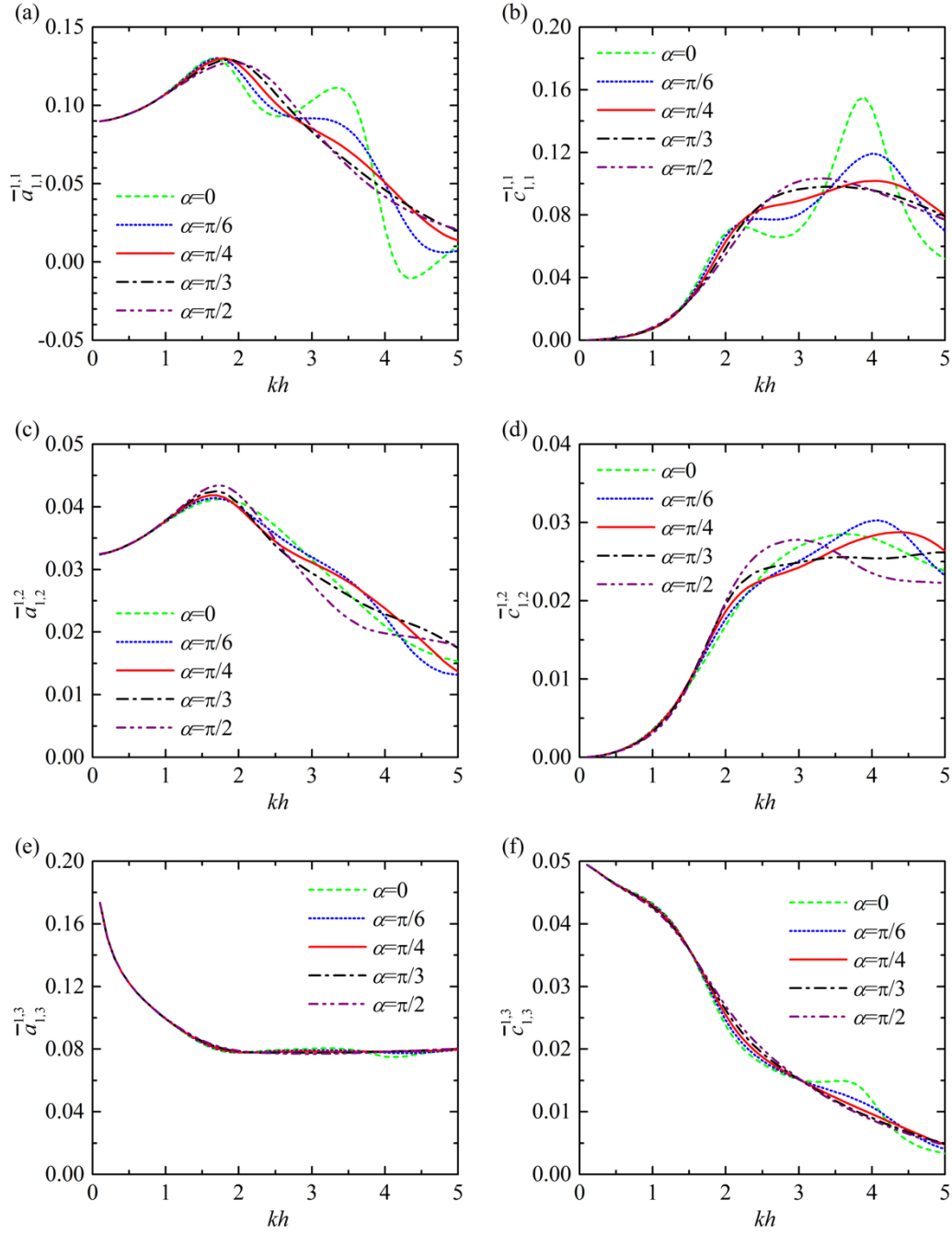
As α increases from 0 to $\pi/2$, as shown in Figs.8a and 8b, the changing trend of $\bar{a}_{1,1}^{1,1} / \bar{c}_{1,1}^{1,1}$ for different kh can be dramatically different and even opposite. For $2.0 < kh < 2.6$, the larger the α , the larger the $\bar{a}_{1,1}^{1,1}$. Whereas as α increases from 0 to $\pi/2$, the value of $\bar{a}_{1,1}^{1,1}$ decreases at $kh=1.5$ and 3.5. In Fig. 8b, opposite effects of α on $\bar{c}_{1,1}^{1,1}$ are observed for $1.6 < kh < 2.1$ and $2.6 < kh < 3.3$.

Figure 8c illustrates that as α increases from 0 to $\pi/2$, the peak value of $\bar{a}_{1,2}^{1,2}$ turns larger and larger, and the corresponding kh is always around 1.7. For kh ranging from 2.8 to 4.1, $\bar{a}_{1,2}^{1,2}$ for $\alpha=\pi/2$ is always smaller than in the other four cases. The frequency response curves of $\bar{c}_{1,2}^{1,2}$ as presented in Fig.8d show obvious fluctuations. The peak value of $\bar{c}_{1,2}^{1,2}$ for different α generally occurs at different wave conditions, e.g., the kh corresponding to the peak of $\bar{c}_{1,2}^{1,2}$ - kh are 3.7, 4.1, 4.4 and 3.0, respectively, for $\alpha=0, \pi/6, \pi/4$ and $\pi/2$.

The effect of α on $\bar{a}_{1,3}^{1,3}$ is found to be rather limited for $0 < kh < 5.0$ (Fig. 8e). Although for most wave frequencies, $\bar{c}_{1,3}^{1,3}$ changes more or less with the change of α , for some specified frequencies, e.g., $kh=1.5$ and 3.0, $\bar{c}_{1,3}^{1,3}$ is found independent of α .

The results of $\bar{a}_{1,4}^{1,4}$ (Fig. 9a) show that there is almost no effect of α on $\bar{a}_{1,4}^{1,4}$ regardless of the oscillating frequency of the caisson. The influence of α on $\bar{c}_{1,4}^{1,4}$ mainly appears at $1.6 < kh < 2.5$ and $kh > 4.2$, for which $\bar{c}_{1,4}^{1,4}$ with $\alpha=\pi/2$ is always the largest one among the five cases (Fig.9b).

The curves of $\bar{a}_{1,5}^{1,5}$ and $\bar{c}_{1,5}^{1,5}$ vs. kh (Figs. 9c and 9d) are found to be similar to those of $\bar{a}_{1,1}^{1,1}$ and $\bar{c}_{1,1}^{1,1}$ (see Figs. 8a and 8b). For all the five cases with different values of α and $0 < kh < 5.0$, as given in Fig. 9e, the maximum value of $\bar{a}_{1,6}^{1,6}$ is 3.7×10^{-3} , occurring at $kh=4.2$ in the case of $\alpha=\pi/6$. $\bar{c}_{1,6}^{1,6}$ in the same case is also observed to be larger than in the rest of the cases for $4.4 < kh < 5.0$ (see Fig. 9f).



1

2 Fig. 8. Normalised hydrodynamic coefficients regarding translational modes vs. kh for different
 3 deployment angle $\alpha=0, \pi/6, \pi/4, \pi/3$ and $\pi/2$: (a) $\bar{a}_{1,1}^{1,1}$; (b) $\bar{c}_{1,1}^{1,1}$; (c) $\bar{a}_{1,2}^{1,2}$; (d) $\bar{c}_{1,2}^{1,2}$; (e) $\bar{a}_{1,3}^{1,3}$; (f)

4 $\bar{c}_{1,3}^{1,3}$.

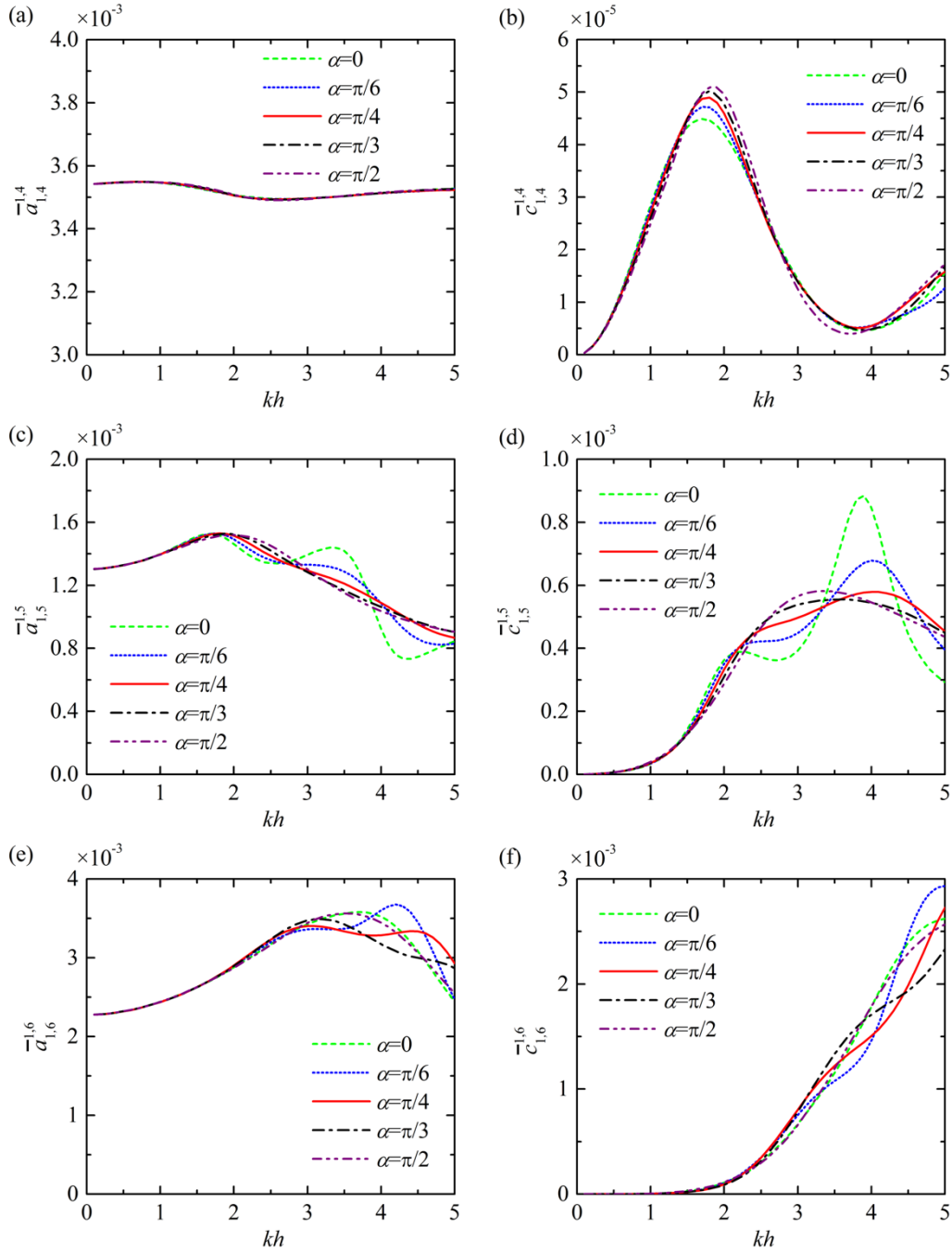


Fig. 9. Normalised hydrodynamic coefficients regarding rotational modes vs kh for different deployment angle $\alpha=0, \pi/6, \pi/4, \pi/3$ and $\pi/2$: (a) $\bar{a}_{1,4}^{-1,4}$; (b) $\bar{c}_{1,4}^{-1,4}$; (c) $\bar{a}_{1,5}^{-1,5}$; (d) $\bar{c}_{1,5}^{-1,5}$; (e) $\bar{a}_{1,6}^{-1,6}$; (f) $\bar{c}_{1,6}^{-1,6}$.

6 Conclusions

The present research focused on wave radiation from multiple truncated cylinders with arbitrary cross sections. A semi-analytic model was developed to investigate the radiation problem based on linear potential theory. The Fourier-series expansion method combined with the eigenfunction expansion matching method was applied to solve the unknown coefficients in the expressions of the radiated spatial potentials.

This semi-analytic model was validated by comparison with a BEM-based numerical model in

a case study: two caissons with the same dimensions deployed side by side. Excellent agreement was achieved between the semi-analytic and the numerical results, demonstrating that the present semi-analytic model can successfully solve the problem of wave radiation from multiple floating cylinders of arbitrary cross sections. Upon validation, the semi-analytic model was applied to investigate the effect of the spacing and the deployment angle on the hydrodynamic coefficients (added-mass and wave radiation damping) of the caissons.

It is found that the hydrodynamic interaction between the caissons can affect the coefficients (relative to the isolated caisson case) depending on the frequency. For this reason, the curves representing the frequency responses of $\bar{a}_{1,n}^{1,n}$ and $\bar{c}_{1,n}^{1,n}$ ($n=1,3,5$) oscillate around the curve of the isolated caisson case. As the spacing between the two caissons increases, the alternation between strengthening and weakening effects of the hydrodynamic interaction occurs more often for the same interval of kh , but the effects themselves gradually wane.

Five configurations of the two caissons with the deployment angle α ranging from 0 to $\pi/2$ are examined. As α increases from 0 to $\pi/2$, the trend of $\bar{a}_{1,1}^{1,1}/\bar{c}_{1,1}^{1,1}$ for different frequencies can be dramatically different, and even opposite. Although for most wave frequencies $\bar{c}_{1,3}^{1,3}$ changes more or less with α , for certain frequencies, e.g., $kh=1.5$ and 3.0 , $\bar{c}_{1,3}^{1,3}$ is found to be independent of α . The influence of α on $\bar{c}_{1,4}^{1,4}$ is apparent mainly for $1.6 < kh < 2.5$ and for $kh > 4.2$, for which $\bar{c}_{1,4}^{1,4}$ with $\alpha=\pi/2$ is always the largest of the five cases.

In sum, the semi-analytic model presented in this paper can be employed to assess the hydrodynamic coefficients of multiple truncated cylinders under small amplitude waves. Together with the semi-analytic model for solving wave diffraction by an array of truncated cylinders with arbitrary cross sections, which will be reported elsewhere, the method can be employed to evaluate the response of many offshore structures, e.g., mobile offshore bases, offshore floating bridges, wave farms and floating offshore wind farms.

Acknowledgements

This research was supported by China Postdoctoral Science Foundation (Grant No. 2016M601041, 2017T100085), National Natural Science Foundation of China (51679124, 51879144) and Intelligent Community Energy (ICE), INTERREG V FCE, European Commission (Contract No. 5025). The first author wishes to thank Dr. Jiabin Liu from Harbin Institute Technology for valuable discussions.

1 **Appendix A. Expression of $f_{\tau,\xi}^{(2),n,i}$**

$$\begin{aligned}
 & f_{\tau,\xi}^{(2),n,i} = \begin{cases} I_{n,\xi}^{(3)} f_{\tau}^{1,n}, & i=1 \\ I_{n,\xi}^{(3)} f_{\tau}^{2,n}, & i=2 \\ -\frac{I_{n,\xi}^{(1)}}{2(h-d_n)} f_{3,0,\tau}^{R,n}, & i=3 \\ -\left(I_{n,\xi}^{(4)} - z_n I_{n,\xi}^{(3)}\right) f_{\tau}^{2,n} + \frac{I_{n,\xi}^{(2)} f_{\tau}^{2,n}}{2(h-d_n)} - \frac{I_{n,\xi}^{(1)} f_{\tau}^{3,n}}{8(h-d_n)}, & i=4 \\ \left(I_{n,\xi}^{(4)} - z_n I_{n,\xi}^{(3)}\right) f_{\tau}^{1,n} - \frac{I_{n,\xi}^{(2)} f_{\tau}^{1,n}}{2(h-d_n)} + \frac{I_{n,\xi}^{(1)} f_{\tau}^{4,n}}{8(h-d_n)}, & i=5 \\ I_{n,\xi}^{(3)} f_{\tau}^{5,n}, & i=6 \end{cases}, \quad (\text{A } 1)
 \end{aligned}$$

3 where $f_{\tau}^{1,n}$, $f_{\tau}^{2,n}$, $f_{\tau}^{3,n}$, $f_{\tau}^{4,n}$, $f_{\tau}^{5,n}$ are the Fourier coefficients at the right hand side of Eqs.

4 (A 2a)~(A 2e) below and they can be calculated by using Eq.(25).

$$\begin{aligned}
 & r_n \left(r_n \cos \theta_n - \frac{\partial S_n}{\partial \theta_n} \sin \theta_n \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_q^{1,n} e^{iq\theta_n}, \quad (\text{A } 2a)
 \end{aligned}$$

$$\begin{aligned}
 & r_n \left(r_n \sin \theta_n + \frac{\partial S_n}{\partial \theta_n} \cos \theta_n \right) \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_q^{2,n} e^{iq\theta_n}, \quad (\text{A } 2b)
 \end{aligned}$$

$$\begin{aligned}
 & r_j^3 \left(3r_j \sin \theta_j + \frac{\partial S_j}{\partial \theta_j} \cos \theta_j \right) \Big|_{S_j=0} = \sum_{q=-\infty}^{\infty} f_q^{3,j} e^{iq\theta_j}, \quad (\text{A } 2c)
 \end{aligned}$$

$$\begin{aligned}
 & r_j^3 \left(3r_j \cos \theta_j - \frac{\partial S_j}{\partial \theta_j} \sin \theta_j \right) \Big|_{S_j=0} = \sum_{q=-\infty}^{\infty} f_q^{4,j} e^{iq\theta_j}, \quad (\text{A } 2d)
 \end{aligned}$$

$$\begin{aligned}
 & r_n^2 \frac{\partial S_n}{\partial \theta_n} \Big|_{S_n=0} = \sum_{q=-\infty}^{\infty} f_q^{5,n} e^{iq\theta_n}, \quad (\text{A } 2e)
 \end{aligned}$$

10 $I_{n,\xi}^{(1)}$, $I_{n,\xi}^{(2)}$, $I_{n,\xi}^{(3)}$ and $I_{n,\xi}^{(4)}$ are expressed as

$$\begin{aligned}
 & I_{n,\xi}^{(1)} = \int_{-h}^{-d_n} \frac{Z_{\xi}(z)}{h} dz = \begin{cases} \frac{N_0^{-0.5} \sinh[k_0(h-d_n)]}{k_0 h}, & \xi=0 \\ \frac{N_{\xi}^{-0.5} \sin[k_{\xi}(h-d_n)]}{k_{\xi} h}, & \xi=1,2,3,\dots \end{cases}, \quad (\text{A } 3a)
 \end{aligned}$$

$$I_{n,\xi}^{(2)} = \int_{-h}^{-d_n} \frac{Z_\xi(z)}{h} (z+h)^2 dz$$

$$= \begin{cases} \frac{N_0^{-0.5}}{k_0^3 h} \left\{ k_0^2 (h-d_n)^2 \sinh[k_0(h-d_n)] - 2k_0(h-d_n) \cosh[k_0(h-d_n)] + 2 \sinh[k_0(h-d_n)] \right\}, & \xi = 0 \\ \frac{N_\xi^{-0.5}}{k_\xi^3 h} \left\{ k_\xi^2 (h-d_n)^2 \sin[k_\xi(h-d_n)] + 2k_\xi(h-d_n) \cos[k_\xi(h-d_n)] - 2 \sin[k_\xi(h-d_n)] \right\}, & \xi = 1, 2, 3, \dots \end{cases}, \quad (\text{A } 3\text{b})$$

$$I_{n,\xi}^{(3)} = \int_{-d_n}^0 \frac{Z_\xi(z)}{h} dz = \begin{cases} \frac{N_0^{-0.5} \left\{ \sinh(k_0 h) - \sinh[k_0(h-d_n)] \right\}}{k_0 h}, & \xi = 0 \\ \frac{N_\xi^{-0.5} \left\{ \sin(k_\xi h) - \sin[k_\xi(h-d_n)] \right\}}{k_\xi h}, & \xi = 1, 2, 3, \dots \end{cases}, \quad (\text{A } 3\text{c})$$

$$I_{n,\xi}^{(4)} = \int_{-d_n}^0 \frac{Z_\xi(z)}{h} z dz = \begin{cases} \frac{N_0^{-0.5}}{k_0^2 h} \left\{ k_0 d_n \sinh[k_0(h-d_n)] - \cosh(k_0 h) + \cosh[k_0(h-d_n)] \right\}, & \xi = 0 \\ \frac{N_\xi^{-0.5}}{k_\xi^2 h} \left\{ k_\xi d_n \sin[k_\xi(h-d_n)] + \cos(k_\xi h) - \cos[k_\xi(h-d_n)] \right\}, & \xi = 1, 2, 3, \dots \end{cases}$$

(A 3d)

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