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# Numerical computations of a melting glass convection in the furnace

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A model of the melting glass convection in a furnace is computed by a finite element method based on the scheme mathematically justified. Some numerical results are demonstrated to investigate differences of the energy efficiency among some electrode configurations.

**Keywords:** melting glass, thermal convection, temperature-dependent coefficient, finite element method

**AMS Subject classification:** 76M10, 76R10, 65M60

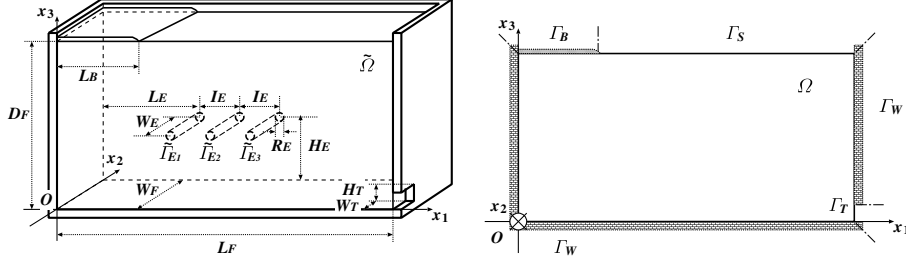
## 1. Introduction

The design for glass furnaces with lower fuel consumption and with less CO<sub>2</sub> emission requires us to analyze precisely convection phenomena in the melting process. For a grasp of phenomena, the use of numerical computations is growing in acceptance, and there are many researches on the computations; see, for example, Choudhary [1], Curran [2], and Ungan and Viskanta [7]. Because the physical properties of raw glass materials depend on the temperature, physical coefficients should be treated with care in devising numerical schemes. Recently, we have mathematically justified a class of finite element methods for thermal convection problems with constant coefficients (see Tagami [5] and Tagami and Itoh [6]) or with temperature-dependent coefficients (see Tabata [3] and Tabata and Tagami [4]). In this paper we apply the scheme to numerical computations of the melting glass convection in the furnace.

Owing to the Boussinesq, the Rosseland (see Viskanta and Anderson [8]), and the infinite Prandtl number approximations, the raw material in the furnace is governed by thermal convection problems with the Joule heat. The equations are discretized by the backward Euler method in time and by the conforming finite elements in space. Temperature-dependent coefficients are treated by an approximation developed in [3] and [4]. Using the numerical scheme, we compute thermal convection phenomena in a model furnace with the Joule heat. We investigate differences among some electrode configurations on the temperature distribution, the convection patterns, and the energies given to the raw materials and passing through the throat.

## 2. A model problem

Figure 1 shows a model furnace to be considered, and its dimensions are listed in Table 1. This model is obtained by normalizing with its depth and rounding one given in [7]. Figure 1(a) shows



(a) A domain  $\tilde{\Omega}$ : a half of the furnace.

(b) A domain  $\Omega$ : the cross section on  $x_2 = 0$  of the furnace.

Figure 1. A model furnace.

Table 1  
The dimensions of the model furnace.

Physical quantity	Symbol	Dimension
Depth of Furnace	$D_F$	1
Length of Furnace	$L_F$	8
Width of Furnace	$W_F$	1
Height of Throat	$H_T$	0.3
Width of Throat	$W_T$	0.4
Radius of Electrodes	$R_E$	0.1
Width of Electrodes	$W_E$	0.5
Height of Electrodes	$H_E$	0.4
Interval between Electrodes	$I_E$	0.8
Length between Back Wall and Electrodes	$L_E$	3
Length of Batch	$L_B$	1

$\tilde{\Omega}$ , a half of the furnace, and Figure 1(b) shows  $\Omega$ , the cross section on  $x_2 = 0$  of the furnace. A raw material is fed into the furnace from the batch  $\Gamma_B$ , and flows out from the throat  $\Gamma_T$ . There is an air-fuel fired boundary  $\Gamma_S$  on the top of the furnace, and when an electric current flows between the electrode  $\tilde{I}_{E1}$ ,  $\tilde{I}_{E2}$ , and  $\tilde{I}_{E3}$ , the Joule heat is generated in the furnace; for the simplicity, the electric current is assumed to be a direct current. The wall  $\Gamma_W$  is insulated.

The electric field is considered in the three-dimensional domain  $\tilde{\Omega}$ . On the other hand, for the simplicity, the flow and the energy fields are considered only in the two-dimensional domain  $\Omega$ . Thus the motion of raw materials can be written as follows: find the velocity  $u$ , the pressure  $p$ , the temperature

$\theta$ , and the electric potential  $\phi$ ,

$$\begin{aligned} (u, p, \theta) &: \Omega \times (0, T) \rightarrow \mathbb{R}^2 \times \mathbb{R} \times \mathbb{R}, \\ \phi &: \tilde{\Omega} \times (0, T) \rightarrow \mathbb{R}, \end{aligned}$$

such that

$$\begin{cases} -\nabla \cdot [\nu(\theta)D(u)] + \nabla p - \beta(\theta)\theta = 0 & \text{in } \Omega \times (0, T), & (2.1a) \\ \nabla \cdot u = 0 & \text{in } \Omega \times (0, T), & (2.1b) \\ \partial_t \theta + (u \cdot \nabla)\theta - \nabla \cdot (\kappa(\theta)\nabla\theta) = \frac{1}{2}\sigma(\theta)|\nabla\phi|^2 & \text{in } \Omega \times (0, T), & (2.1c) \\ -\nabla \cdot (\tilde{\sigma}(\theta)\nabla\phi) = 0 & \text{in } \tilde{\Omega} \times (0, T) & (2.1d) \end{cases}$$

with the boundary conditions for the flow field

$$\begin{cases} u = u_D & \text{on } \Gamma_W \cup \Gamma_B \times (0, T), & (2.2a) \\ \tau(u, p, \theta) = 0 & \text{on } \Gamma_T \times (0, T), & (2.2b) \\ u \cdot n = 0, \quad \tau(u, p, \theta) \cdot t^{(1)} = 0 & \text{on } \Gamma_S \times (0, T), & (2.2c) \end{cases}$$

for the energy field

$$\begin{cases} \theta = \theta_D & \text{on } \Gamma_B \cup \Gamma_S \times (0, T), & (2.3a) \\ \frac{\partial \theta}{\partial n} = 0 & \text{on } \Gamma_T \cup \Gamma_W \times (0, T), & (2.3b) \end{cases}$$

and for the electric field

$$\begin{cases} \phi = \phi_D & \text{on } \tilde{\Gamma}_E \times (0, T), & (2.4a) \\ \frac{\partial \phi}{\partial n} = 0 & \text{otherwise,} & (2.4b) \end{cases}$$

and with the initial condition

$$\theta = \theta^0 \quad \text{in } \Omega \text{ at } t = 0, \quad (2.5)$$

where  $\tilde{\Gamma}_E$  denotes a union of the electrode boundaries,  $\tilde{\Gamma}_{E_1} \cup \tilde{\Gamma}_{E_2} \cup \tilde{\Gamma}_{E_3}$ ;  $n$  an outward unit normal, and  $t^{(1)}$  a unit tangent;

$$\begin{aligned} u_D &: \Gamma_W \cup \Gamma_B \times (0, T) \rightarrow \mathbb{R}^2, \\ \theta_D &: \Gamma_B \cup \Gamma_S \times (0, T) \rightarrow \mathbb{R}, \quad \phi_D : \tilde{\Gamma}_E \times (0, T) \rightarrow \mathbb{R} \end{aligned}$$

a set of boundary velocity, temperature, and electric potential;

$$\theta^0 : \Omega \rightarrow \mathbb{R}$$

an initial temperature;

$$(\nu, \kappa, \sigma, \beta) : \Omega \times (0, T) \times \mathbb{R} \rightarrow \mathbb{R}^+ \times \mathbb{R}^+ \times \mathbb{R}^+ \times \mathbb{R}^2$$

a set of viscosity, thermal conductivity, electrical conductivity, and thermal expansion coefficient depending on  $x$ ,  $t$ , and  $\theta$ ;  $\tau(u, p, \theta)$  the surface stress defined by

$$\tau(u, p, \theta) \equiv \{-pI + 2\nu(\theta)D(u)\} \cdot n$$

with the identity tensor  $I$ ; the value of  $\overline{\nabla\phi}$  in (2.1c) is defined by

$$\overline{\nabla\phi}(x_1, x_3, t) \equiv \left( \frac{\partial\phi}{\partial x_1}(x_1, 0, x_3, t), \quad 0, \quad \frac{\partial\phi}{\partial x_3}(x_1, 0, x_3, t) \right)^T;$$

moreover,  $\tilde{\sigma}$  is defined by  $\tilde{\sigma}(x_1, x_2, x_3, t) \equiv \sigma(x_1, x_3, t)$ .

### 3. Finite element approximation

Let  $X$ ,  $Y$ ,  $Q$ , and  $\tilde{X}$  be function spaces defined by  $X \equiv H^1(\Omega)$ ,  $Y \equiv X^2$ ,  $Q \equiv L^2(\Omega)$ , and  $\tilde{X} \equiv H^1(\tilde{\Omega})$ , respectively; let  $(\cdot, \cdot)$  denote the inner product of  $L^2(\Omega)$ ; let  $\Psi(\theta)$ ,  $V(u)$ , and  $\Xi(\phi)$  be function spaces defined by

$$\begin{aligned} \Psi(\theta) &\equiv \{\psi \in X; \psi = \theta \text{ on } \Gamma_B \cup \Gamma_S\}, \\ V(u) &\equiv \{v \in Y; v = u \text{ on } \Gamma_W \cup \Gamma_B, v \cdot n = 0 \text{ on } \Gamma_S\}, \\ \Xi(\phi) &\equiv \{\xi \in \tilde{X}; \xi = \phi \text{ on } \tilde{\Gamma}_E\}, \end{aligned}$$

respectively; and let  $\Psi$ ,  $V$ , and  $\Xi$  be function spaces defined by  $\Psi \equiv \Psi(0)$ ,  $V \equiv V(0)$ , and  $\Xi \equiv \Xi(0)$ , respectively. We prepare bi- and tri-linear forms  $a_0$ ,  $b$ ,  $c_0$ , and  $c_1$  defined by

$$\begin{aligned} a_0(\nu; u, v) &\equiv 2 \int_{\Omega} \nu D(u) : D(v) \, dx && \text{for } (\nu, u, v) \in L^\infty(\Omega) \times Y \times Y, \\ b(v, q) &\equiv - \int_{\Omega} q \nabla \cdot v \, dx && \text{for } (v, q) \in Y \times Q, \\ c_0(\kappa; \theta, \psi) &\equiv \int_{\Omega} \kappa \nabla \theta \cdot \nabla \psi \, dx && \text{for } (\kappa, \theta, \psi) \in L^\infty(\Omega) \times X \times X, \\ c_1(w, \theta, \psi) &\equiv \int_{\Omega} (w \cdot \nabla \theta) \psi \, dx && \text{for } (w, \theta, \psi) \in Y \times X \times X. \end{aligned}$$

We also prepare a bilinear form  $\tilde{c}_0$  by replacing the integral region  $\Omega$  of  $c_0$  by  $\tilde{\Omega}$ .

We decompose the half furnace  $\tilde{\Omega}$  into a union of tetrahedra, and consider its cross section on  $x_2 = 0$  as the triangulation of  $\Omega$ . We introduce finite dimensional spaces  $X_h$ ,  $Q_h$ , and  $\tilde{X}_h$  approximating  $X$ ,  $Q$ , and  $\tilde{X}$ , respectively. Let  $\Psi_h(\theta)$  be a subspace of  $X_h$  approximating  $\Psi(\theta)$ ; let  $V_h(u)$  be a subspace of  $Y_h \equiv X_h^2$  approximating  $V(u)$ ; and let  $\Xi_h(\phi)$  be a subspace of  $\tilde{X}_h$  approximating  $\Xi(\phi)$ .

Let  $\tau$  be a time increment and  $N_T \equiv [T/\tau]$  a total step number. The time  $n\tau$  is denoted by  $t_n$ . We denote by  $\theta^n$  the value  $\theta(n\tau)$  at time step  $n$  and by  $\overline{D}_\tau \theta^n$  the backward difference quotient  $(\theta^n - \theta^{n-1})/\tau$ .

Let us denote by  $\nu_h^{n,m} \in Z_h$  the approximate viscosity  $\Pi_h[\nu^n(\theta_h^m)]$ . Similarly,  $\kappa_h^{n,m}$ ,  $\sigma_h^{n,m}$ , and  $\beta_h^{n,m}$  are defined, where  $Z_h$  is a finite dimensional subspace of  $Q$  and  $\Pi_h$  an interpolation to  $Z_h$ .

Hereafter, the boundary data  $u_D$ ,  $\theta_D$ , and  $\phi_D$  are assumed to be independent of time. We introduce an approximate problem discretized by the backward Euler method in time and by the finite element method in space: Setting  $\theta_h^0 \in \Psi_h(\theta_D)$  as an approximation to  $\theta^0$ , we find  $\{(u_h^n, p_h^n, \theta_h^n, \phi_h^n) \in V_h(u_D) \times$

$Q_h \times \Psi_h(\theta_D) \times \Xi_h(\phi_D); n = 1, \dots, N_T\}$  such that for  $n = 1, \dots, N_T$

$$\begin{cases} a_0(\nu_h^{n,n-1}; u_h^n, v_h) + b(v_h, p_h^n) \\ \quad - (\beta_h^{n,n-1} \theta_h^{n-1}, v_h) = 0 & \text{for all } v_h \in V_h, \end{cases} \quad (3.1a)$$

$$\begin{cases} b(u_h^n, q_h) = 0 & \text{for all } q_h \in Q_h, \end{cases} \quad (3.1b)$$

$$\begin{cases} (\overline{D}_\tau \theta_h^n, \psi_h) + c_0(\kappa_h^{n,n-1}; \theta_h^n, \psi_h) \\ \quad + c_1(u_h^{n-1}, \theta_h^n, \psi_h) = \frac{1}{2}(\sigma_h^{n,n-1} |\nabla \phi_h^n|^2, \psi_h) & \text{for all } \psi_h \in \Psi_h, \end{cases} \quad (3.1c)$$

$$\begin{cases} \tilde{c}_0(\tilde{\sigma}_h^{n,n-1}; \phi_h^n, \xi_h) = 0 & \text{for all } \xi_h \in \Xi_h. \end{cases} \quad (3.1d)$$

At  $n$ th time step, the approximate problem is split up into three parts; one is the Stokes part (3.1a) and (3.1b), another the generalized convection-diffusion part (3.1c), and the other the Poisson part (3.1d). Assuming some conditions to choose a pair of finite elements  $(X_h, Q_h, \tilde{X}_h, Z_h)$ , we can prove the solvability of (3.1); for detail, see [4] and [6]. As a result, the solution  $\{(u_h^n, p_h^n, \theta_h^n, \phi_h^n)\}$  is obtained step by step from  $\theta_h^0$ .

#### 4. Numerical results

Set  $T = 0.5$ . The physical coefficients  $\nu$ ,  $\kappa$ ,  $\sigma$ , and  $\beta$  are defined by

$$\begin{aligned} \nu &\equiv 0.01 \exp\left(\frac{50}{10 + \theta}\right), & \kappa &\equiv 0.01 \theta^2 + 0.2 \theta + 1, \\ \sigma &\equiv 55 \exp\left(\frac{-50}{15 + \theta}\right), & \beta &\equiv (0, 20000). \end{aligned}$$

These coefficients are obtained by normalizing with 1500 [K] and by rounding those of a raw glass material given in [7]. For detail of the normalization, see [5]. From this normalization, the values of unknown functions lie roughly in

$$\begin{aligned} -4 \times 10^2 &\lesssim u_1 \lesssim 2 \times 10^3, & -4 \times 10^2 &\lesssim u_2 \lesssim 1 \times 10^3, \\ -3 \times 10^{-6} &\lesssim p \lesssim 5 \times 10^5, & -3 &\lesssim \theta, \phi \lesssim 4. \end{aligned}$$

Figure 2 shows the graph of  $\nu$ ,  $\kappa$ , and  $\sigma$  versus the temperature. Table 2 shows the common control data

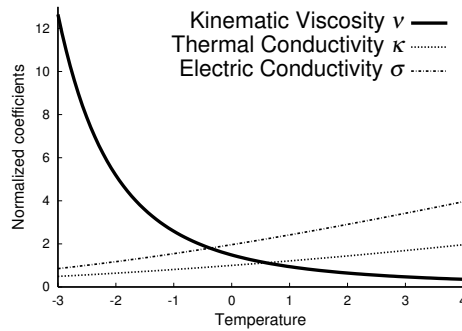


Figure 2. The temperature-dependent coefficients.

for  $u$  and  $\theta$  on the boundary and at the initial time, which are also normalized with 1500 [K]. Table 3

lists three cases of the electric potential values on  $\tilde{\Gamma}_E$ . In each case, the sum of the values is equal to 0. In Case A, the values are obtained by normalizing and rounding the root-mean-square values of the complex potentials given in [7]. For detail of the normalization, see [5] again. In Cases B and C, the locations of the positive potential are different from those in Case A.

Table 2

The common control data for  $u$  and  $\theta$  on the boundary and at the initial time.

Place	$u$	Place	$\theta$	Time	$\theta$
$\Gamma_W$	(0, 0)	$\Gamma_B$	-3	$t = 0$	0
$\Gamma_B$	(400, 8)	$\Gamma_S$	3		

Table 3

The electric potential  $\phi$  imposed on  $\tilde{\Gamma}_E$ .

Case	$\tilde{\Gamma}_{E_1}$	$\tilde{\Gamma}_{E_2}$	$\tilde{\Gamma}_{E_3}$
A	6	-3	-3
B	-3	6	-3
C	-3	-3	6

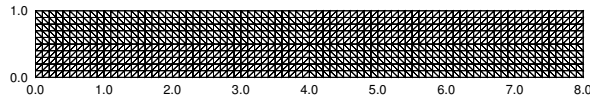


Figure 3. The cross section  $\Omega$  and its triangulation.

The domain  $\tilde{\Omega}$  is divided into a union of  $80 \times 10 \times 10$  cuboids, and each cuboid is divided into 6 tetrahedra, that is, the domain  $\tilde{\Omega}$  is divided into a union of 48,000 tetrahedra. Figure 3 shows the triangulation of  $\Omega$ . We denote the maximum diameter of the triangles by  $h = \sqrt{2}/10 \approx 0.14$ . To approximate the velocity, the pressure, the temperature, and the electric potential, the finite elements  $P2/P1/P2/P2$  are used, which have the second order approximability; see [4]. The physical coefficients are approximated by  $P1$  elements, which is the most efficient choice; see also [4]. The time increment  $\tau$  is set to be  $\tau = 4.0 \times 10^{-3}$  so that the size of  $\tau$  may resolve the period of oscillation phenomena in the following Figure 5.

At each step, the Stokes part (3.1a) and (3.1b) and the generalized convection-diffusion part (3.1c) were solved by the Gauss elimination method, and the Poisson part (3.1d) was solved by the Conjugate Gradient (CG) method with the incomplete  $LU$  factorization preconditioner. The CG method was stopped when its relative residual becomes less than  $10^{-8}$ . The computation was performed by Pentium4 3.2GHz with 2GB memories. In each case, it took about 5 hours to compute.

Figure 4 shows the streamlines, the temperature, and the electric potential at 0.16-time-intervals in Case A. At  $t = 0.02$ , there is a large roll cell under the batch, and the raw material is not yet fully



heated. As the time goes by, there appears a series of roll cells, which become weak in approaching the throat.

Figure 5 shows the history of a Joule heat  $J$  and an energy  $E$  defined by

$$J \equiv \frac{1}{2} \int_{\Omega} \sigma(\theta) |\nabla \phi|^2 dx, \quad E \equiv \int_{\Gamma_T} u_1 \theta ds.$$

In Figure 5(b), for the comparison, there is also the graph in the case without the Joule heat, “Without Rods”. The averages of the energies in Cases A, B, and C are about 1.08 times larger than that in “Without Rods”. The periodic motion of the convection phenomena causes the oscillation of  $J$  and  $E$ . The amplitudes of  $E$  when an electric current flows becomes smaller than those when a current does not; the amplitude of  $E$  in “Without Rods” is about 10–15% of the value of  $E$  itself; on the other hand, those in Cases A, B, and C are about 2–3%. The period of these oscillations is about 0.008–0.012. Figure 6 shows the electric potential on  $x_3 = 0.3$  in Cases A and B. From Figure 6 the gradient of the electric potential on  $x_2 = 0$  in Case B is less than that in Case A. This fact imply, as in Figure 5, the Joule heat in Case B is about half of those in Cases A and C. Figure 7 shows the streamlines and the temperature near the throat. The upper figure shows the result when the oscillation of the energy achieves a trough at  $t = 0.408$ . The lower figure shows the result when the oscillation of the energy achieves a peak at  $t = 0.416$ . The pattern of the streamlines at the peak is distinguished from that at the trough. The center of the nearest role cell to the throat is about  $6.3 \leq x_1 \leq 6.7$  at the trough, and is about  $7.1 \leq x_1 \leq 7.2$  at the peak. The distributions of the temperature, meanwhile, are almost same at both the peak and the trough. These facts imply that the periodic motion of the flow is related to those of  $J$  and  $E$ .

Table 4 shows the Joule heat and the energy consumptions. “Given” and “Out” mean the Joule heat  $G$  on  $x_2 = 0$  and the energy  $O$  passing the throat defined by

$$G \equiv \int_0^T J dx, \quad O \equiv \int_0^T \int_{\Gamma_T} u_1 (\theta - \theta_0) ds,$$

respectively. Here  $\theta_0$  is the temperature in the case without the electrodes. “Ratio” is defined by the ratio of “Out” to “Given”. This table shows that both Joule heat and “Out” energy in Case C are the highest among the three cases. As mentioned in Figure 6, the Joule heat in Case B is about a half of those in the other cases. “Ratio” in Case C is 1.3–1.6 times as high as those in the other cases. As a result, Case C is the best configuration among the cases.

Table 4  
The energy consumptions in each case.

Case	Given	Out	Ratio [%]
A	19.9	1.13	5.70
B	9.3	0.43	4.63
C	20.2	1.47	7.25

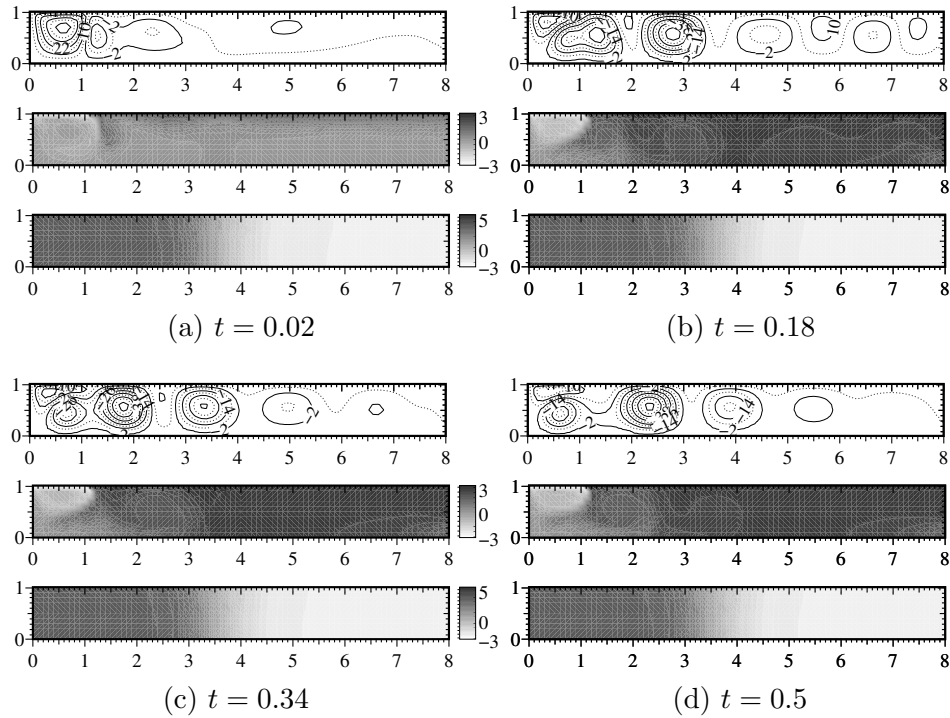


Figure 4. The streamlines (Upper), the temperature (Middle), and the electric potential (Lower) in Case A.

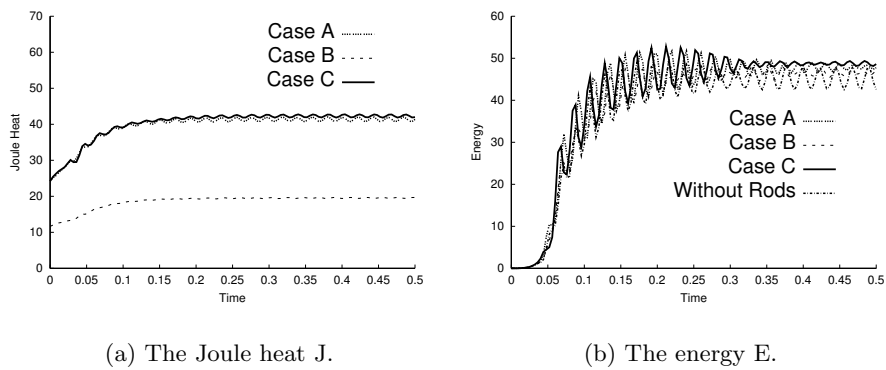


Figure 5. The Joule heat on  $x_2 = 0$  and the energy at the throat.

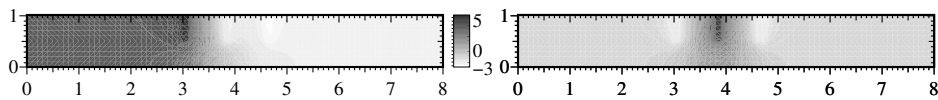


Figure 6. The electric potential in Cases A (left) and B (right) on  $x_3 = 0.3$ .

## 5. Concluding remarks

In this paper we have computed a melting glass convection in a model furnace by a finite element method derived from [3]–[6]. Some numerical results have been demonstrated to investigate differences of the energy efficiency among some electrode configurations.

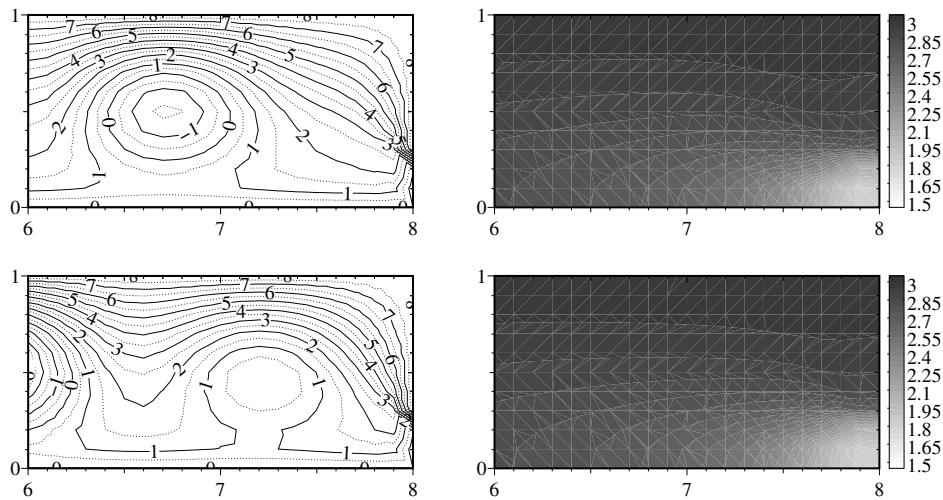


Figure 7. The streamlines (Left) and the temperature (Right) near the throat in Case A at  $t = 0.408$  (Above) and  $0.416$  (Below).

In this paper we simplify a part of the problem and solve the heat convection equations in a cross section. We are going to solve the whole problem in the three-dimensional configuration.

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